

Control in Haptics: Towards Optimal Haptic Feedback

WHC-Workshop: Haptic Methods and Technologies for Virtual Assembly Simulations

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Knowledge for Tomorrow

Motivation

“Haptic rendering is the process of computing and generating forces in response to user interactions with virtual objects”

[Salisbury et al., 1995]



Motivation

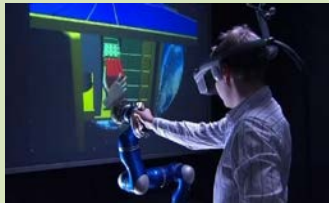
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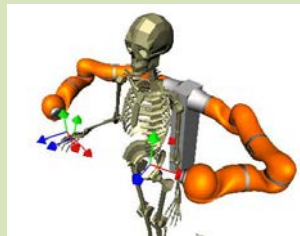
Haptic Rendering Applications



assembly
simulation



training of
mechanics



rehabilitation



augmentation
in telerobotics

ergonomics
analysis,
computer
games,
education,
...



Motivation

“Haptic
respons
[Salisbu

forces in



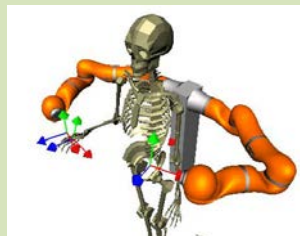
HUG – DLR’s bimanual haptic device
[Hulin et al., ENACTIVE2008], [Hulin et al., ICRA2011]



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augmentation
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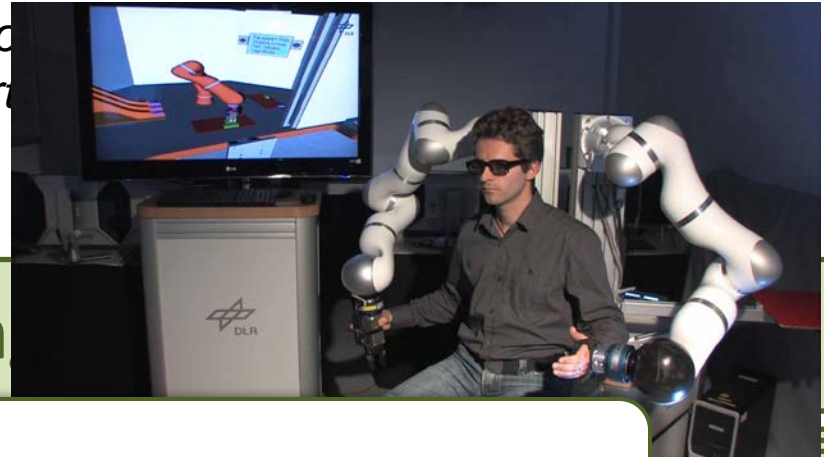
ergonomics
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Motivation

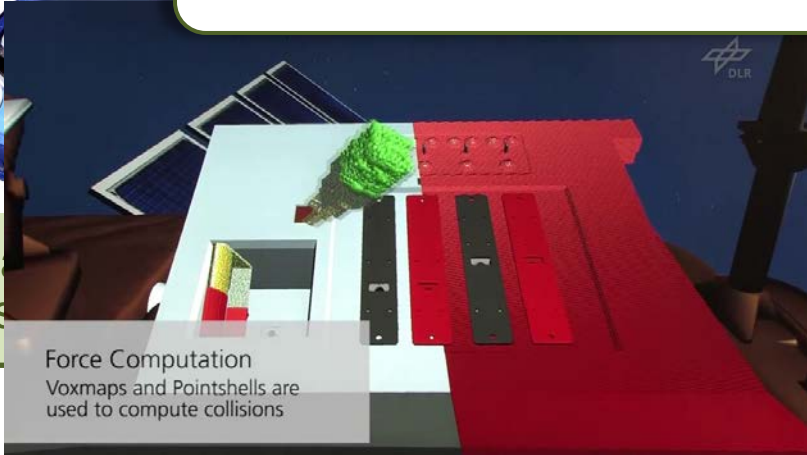


Collision forces (2 x 6-DoF) are displayed via two light-weight robots which cover



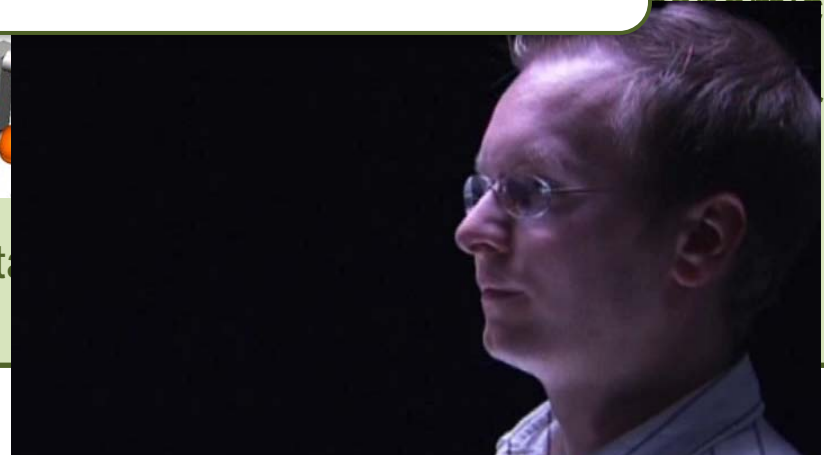
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Fundamental Requirement: Stability!



Force Computation
Voxmaps and Pointshells are used to compute collisions

Stability

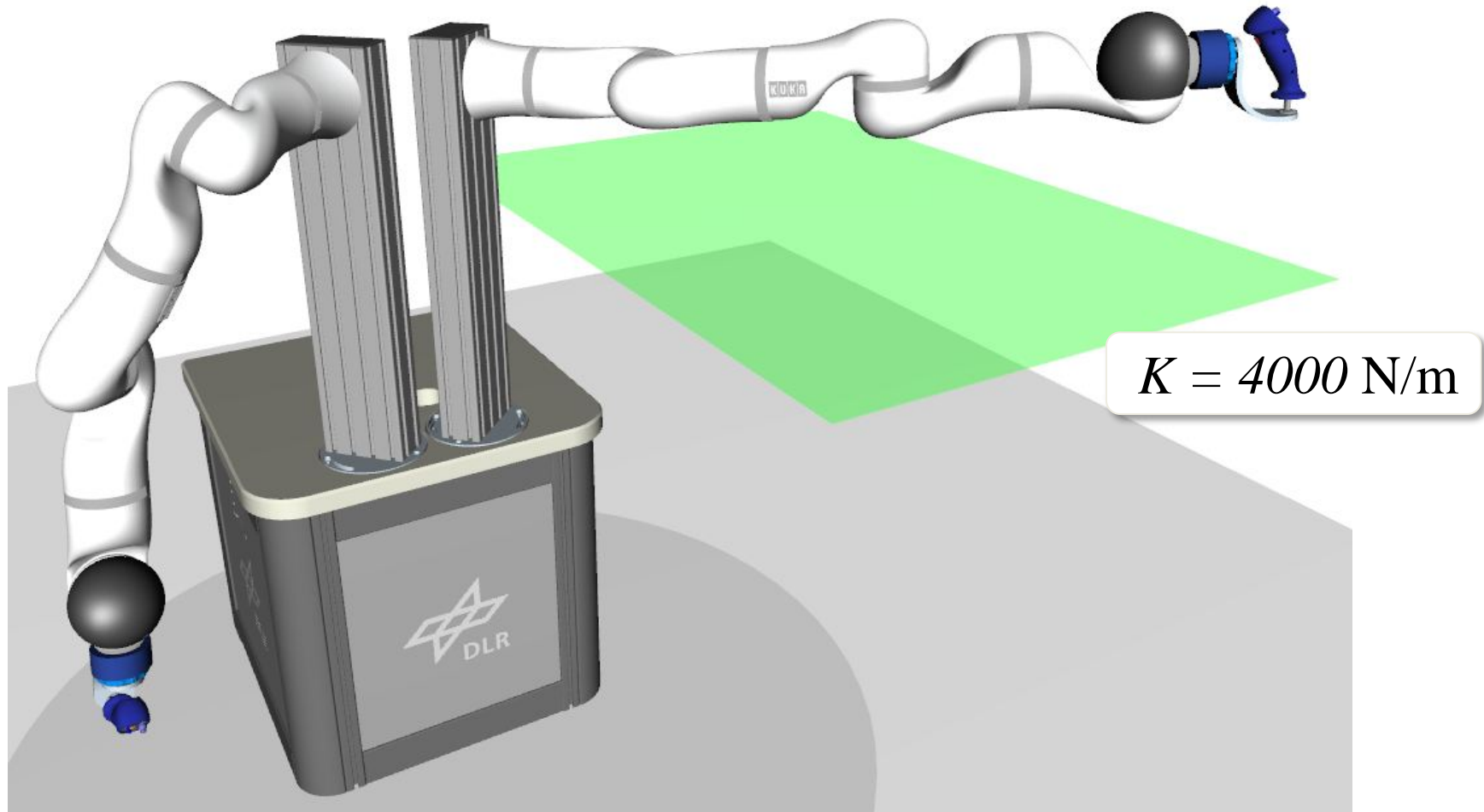


stability



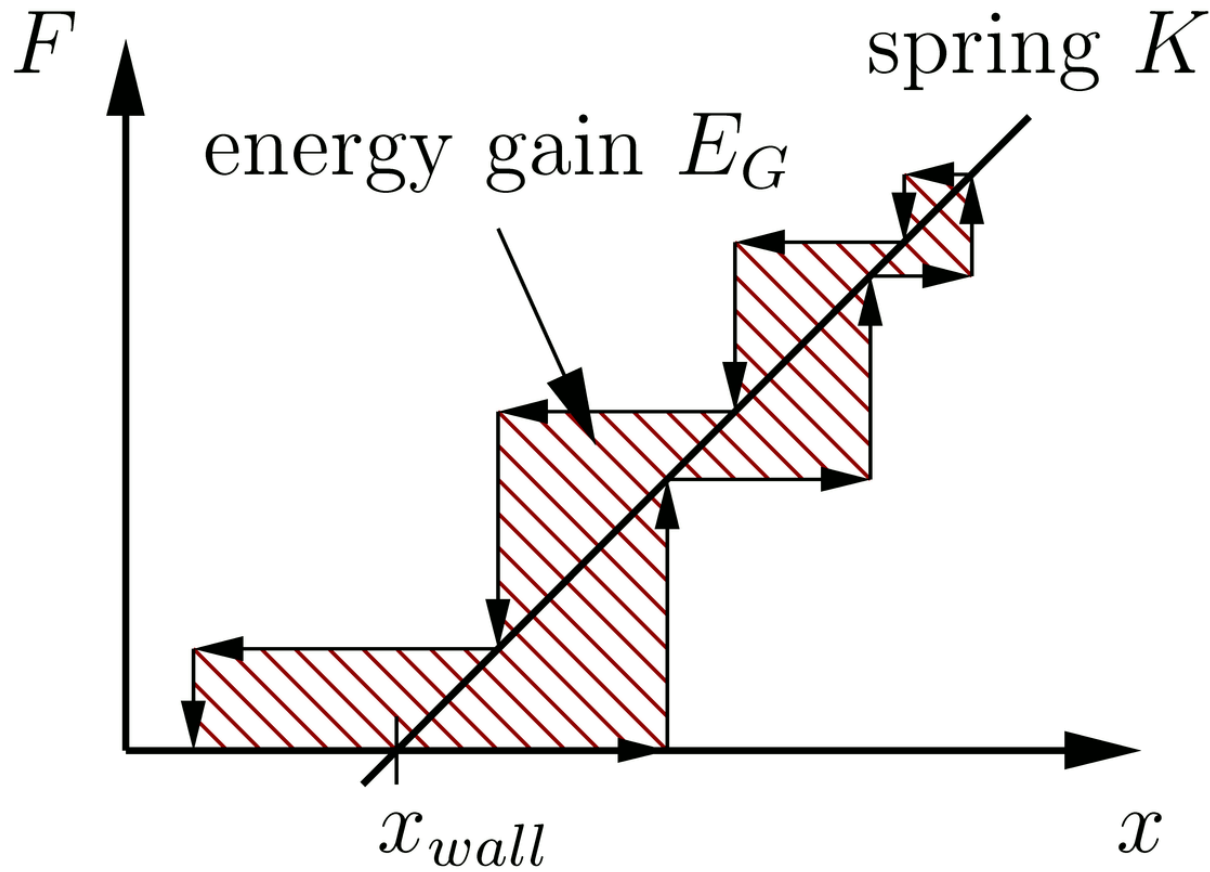
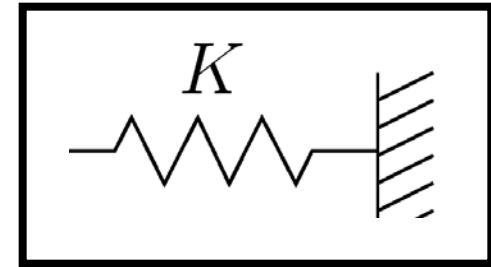
Motivation

Virtual springs are active!



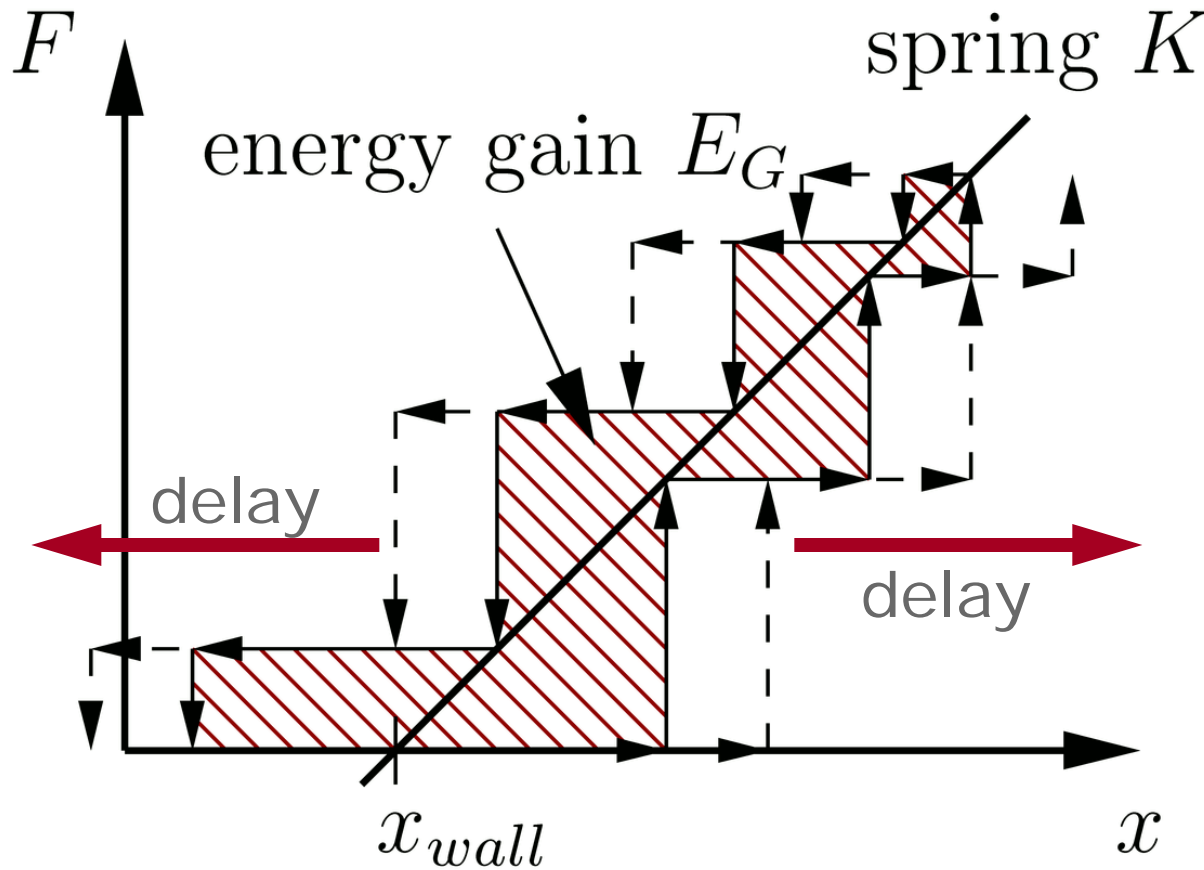
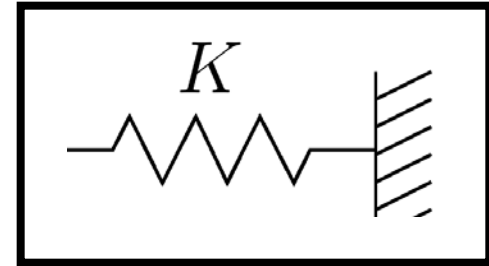
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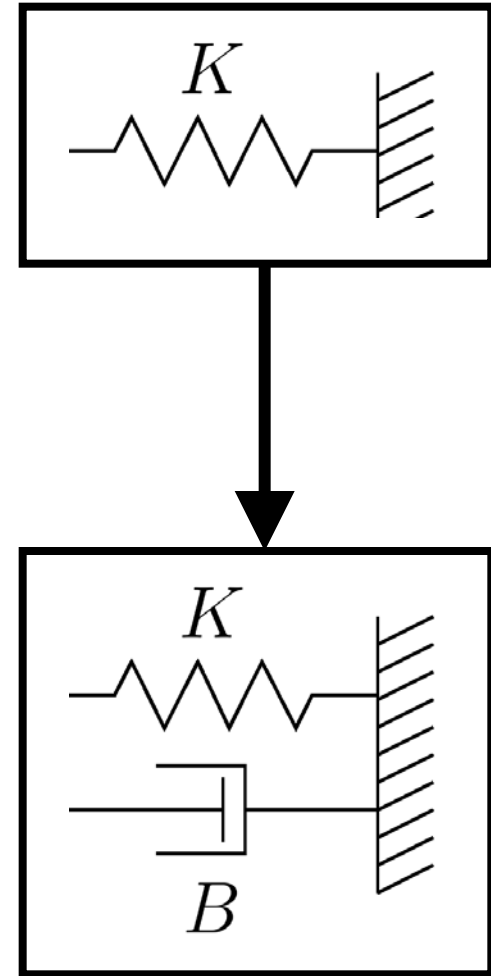
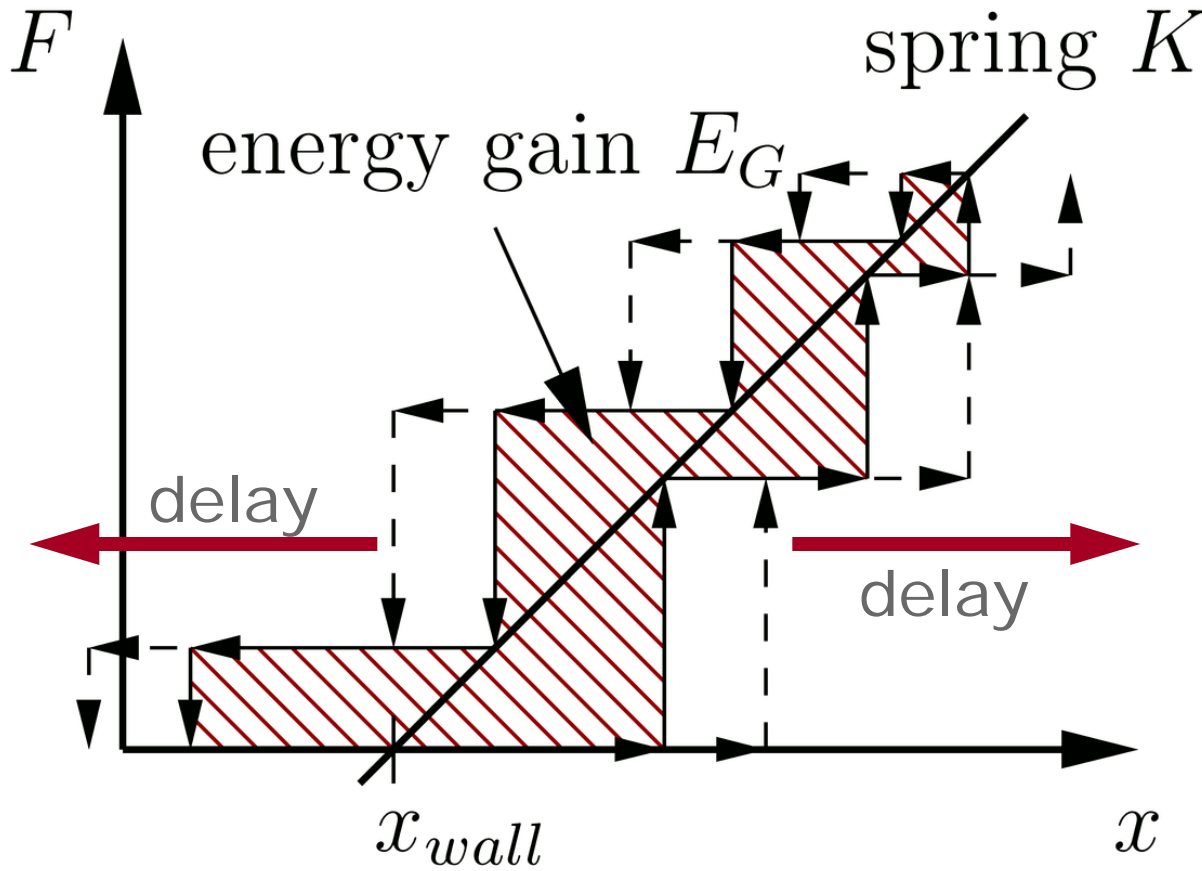
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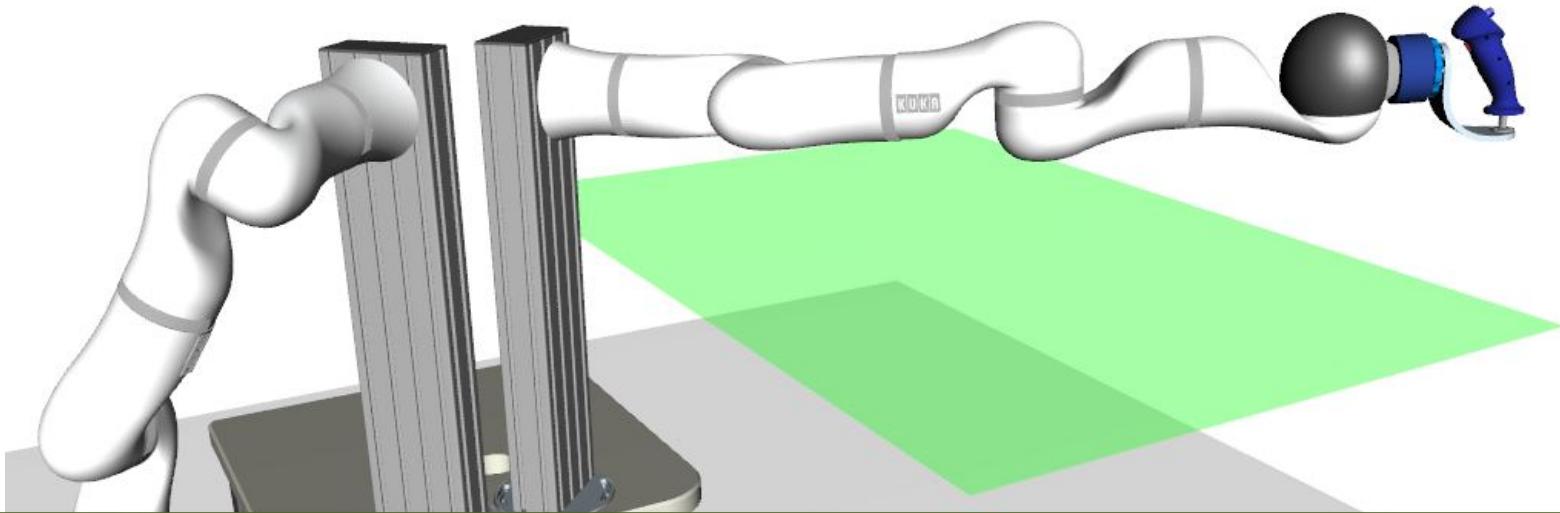
Virtual springs are active!



Motivation

Virtual springs are active!

$$K = 4000 \text{ N/m}$$
$$B = 60 \text{ Ns/m}$$



- For which parameters is the haptic system stable?
- What is the relation between stable and passive parameters?
- What are the optimal wall parameters?
- What is the influence of the human operator?



State of the Research

[Hulin Dissertation]



State of the Research

Table 2.1: Key approaches in control of haptic rendering.

timeline	1990	1994	1996	2000	2002	2003	2005	2006	2007	2008	2015	2014
first author	Minsky	Colgate	Salcudean	Miller	Hannaford	Gil	Abbott	Diolaiti	Iskakov	Diaz	Paine	Hulin
reference	[81]	[23]	[99]	[80]	[42]	[35]	[4]	[33]	[60]	[30]	[90]	[48, 50]
analysis												
stability	+	−	+	−	−	+	−	+	+	+	+	+
passivity	−	+	−	+	+	−	+	+	−	−	+	+
optimal control	−	−	−	−	−	−	−	−	−	−	+	+
linear properties												
m mass	+	+	+	+	−	+	+	+	+	+	+	+
K discrete-time stiffness	\pm^a	+	+	+	\pm^b	+	+	+	+	+	\pm^a	+
B discrete-time damping	\pm^a	+	+	+	+	+	−	−	+	−	\pm^a	+
b viscous friction	+	+	−	+	−	+	+	+	+	+	+	+
human operator	+	\pm^c	−	\pm^c	\pm^c	+	\pm^c	−	−	−	−	+
d delay	+	−	\pm^d	\pm^d	\pm^d	−	−	+	−	−	+	+
velocity filtering	−	+	−	−	−	−	−	−	−	−	+	−
structural elasticities	−	−	−	−	−	−	−	−	−	+	−	−
nonlinear properties												
unilateral wall	−	+	−	+	−	−	+	−	−	−	−	\pm^e
c static friction	−	−	+	−	−	−	+	+	+	−	−	−
Δ pos. sensor quantization	−	−	−	−	−	−	+	+	+	−	−	−

^aThe effect of time-discretization was only considered within numerical simulations.

^bThe time-domain passivity approach does not require any assumption on the virtual environment but it takes into account the effect of time-discretization, as it assumes a constant force between two sampling instants.

^cThe passivity analysis is independent of a human operator grasping the haptic device. Thus, although not investigated directly, this work takes into account humans operators.

^dA constant delay of up to one sampling period was considered.

^eThe unilateral wall is taken into account for passivity and not for stability.

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linear properties												
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b viscous friction	+	+	-	+	-	+	+	+	+	+	+	+
human operator	+	\pm^c	-	\pm^c	\pm^c	+	\pm^c	-	-	-	-	+
d delay	+	-	\pm^d	\pm^d	\pm^d	-	-	+	-	-	+	+
velocity filtering	-	+	-	-	-	-	-	-	-	-	+	-
structural elasticities	-	-	-	-	-	-	-	-	-	+	-	-
nonlinear properties												
unilateral wall	-	+	-	+	-	-	+	-	-	-	-	\pm^e
c static friction	-	-	+	-	-	-	+	+	+	-	-	-
Δ pos. sensor quantization	-	-	-	-	-	-	+	+	+	-	-	-

^aThe effect of time-d

^bThe time-domain p

^cThe passivity analy

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$$b > \frac{KT}{2} + |B|$$

$$c \geq \frac{K\Delta}{2}$$

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human operator	+	\pm^c	-	\pm^c	\pm^c	+	\pm^c	-	-	-	-	+
d delay	+	-	\pm^d	\pm^d	\pm^d	-	-	+	-	-	+	+
velocity filtering	-	+	-	-	-	-	-	-	-	-	+	-
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Outline

- 1. Stability (9 Minutes)**
- 2. Optimal Control (7 Minutes)**
- 3. Experiments (5 Minutes)**



Part 1: Stability

Stability Analysis of Haptic Systems

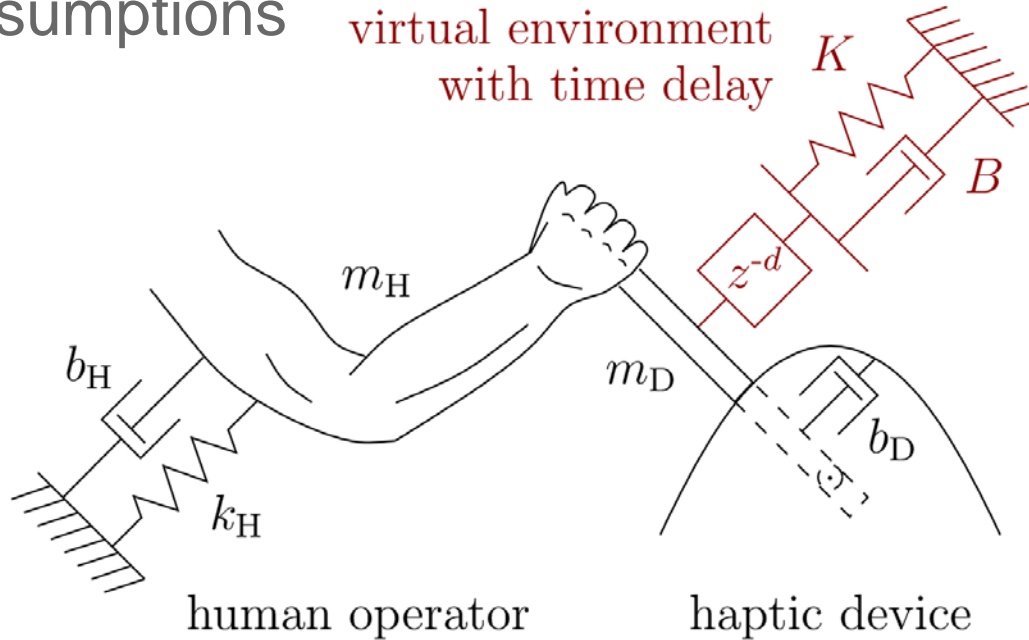
[Hulin et al., SYROCO2006],
[Hulin et al., IROS2006],
[Hulin et al., IROS2008], ...



System Description

Assumptions

virtual environment
with time delay



- 1 DoF
- Linear model of human
- Delay is permitted
- Other nonlinear effects are neglected
- Direct coupling between m_D and m_H

Virtual Wall

K : virtual stiffness

B : virtual damping

T : sampling period

t_d : time delay ($t_d = d \cdot T$)

Haptic Device

b_D : physical damping

m_D : mass

Human

k_H : physical stiffness

b_H : physical damping

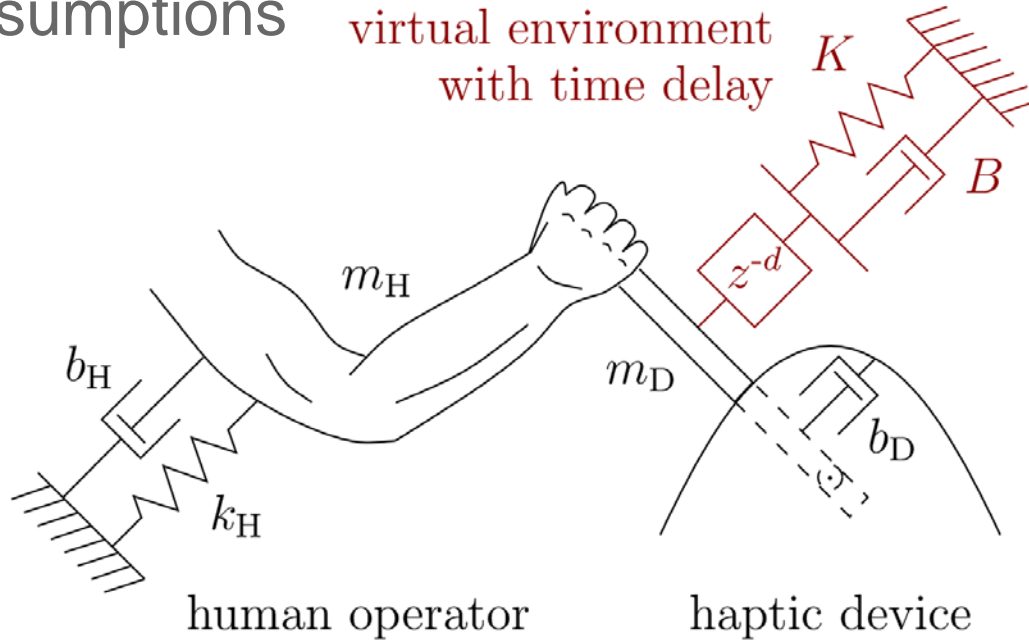
m_H : mass



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Virtual Wall

K : virtual stiffness

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Haptic Device

b_D : physical damping

m_D : mass

Human

k_H : physical stiffness

b_H : physical damping

m_H : mass

$$m = m_D + m_H$$

$$b = b_D + b_H$$

$$k = k_H$$



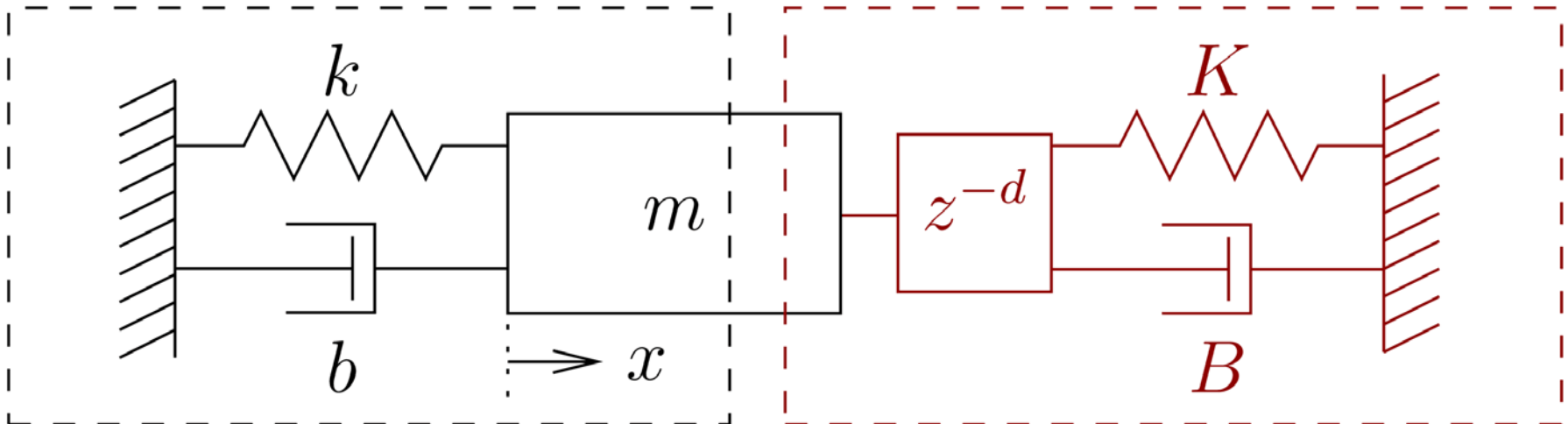
System Description

Assumptions

- K : virtual stiffness
- B : virtual damping
- T : sampling period
- t_d : time delay ($t_d = d \cdot T$)
- k : physical stiffness
- b : physical damping
- m : mass

real world

virtual world



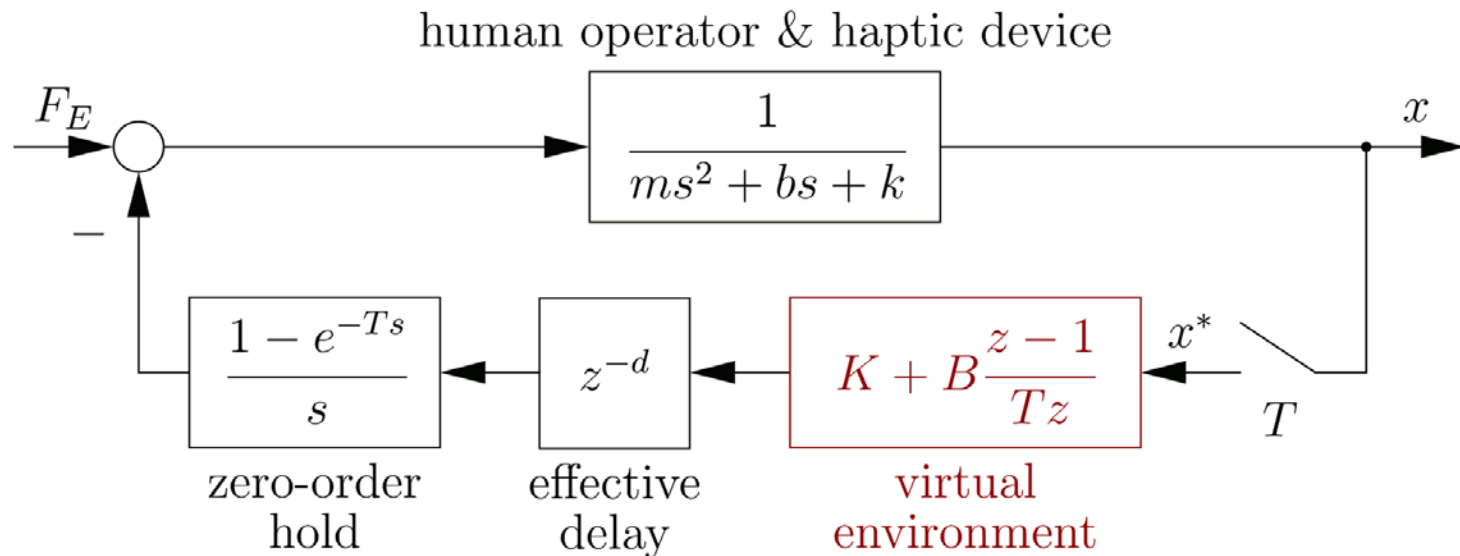
System with 7 parameters



System Description

Control Loop

$$\text{Time Delay} \\ t_d = d \cdot T$$



Consists of continuous- and discrete-time blocks

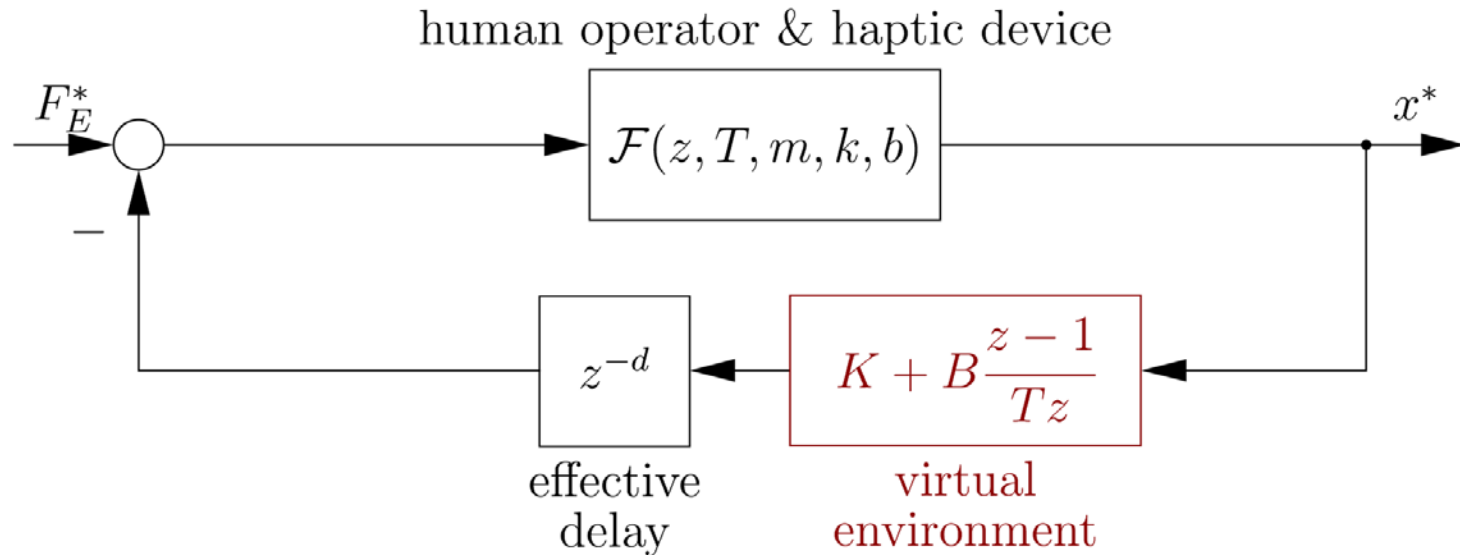
- Use ZOH-Equivalent of continuous-time block
(= **Exact description!**)



System Description

Control Loop

Time Delay
 $t_d = d \cdot T$



$$\mathcal{Z} \{H_x(s)H_0(s)\} = \frac{2(z + e^{-bT/m}) - (z + 1)(c_2 + c_3) + bT(z - 1)(c_2 - c_3)/(mc_1)}{2k(z^2 - (c_2 + c_3)z + e^{-bT/m})}$$

with

$$c_1 = \sqrt{(bT/m)^2 - 4kT^2/m}$$

$$c_2 = e^{-(bT/m + c_1)/2}$$

$$c_3 = e^{-(bT/m - c_1)/2}$$



System Description

Normalization

Time Delay

$$t_d = d \cdot T$$

$\alpha = K \cdot T^2 / m$: normalized virtual stiffness

$\beta = B \cdot T / m$: normalized virtual damping

$\gamma = k \cdot T^2 / m$: normalized physical stiffness

$\delta = b \cdot T / m$: normalized physical damping

Characteristic Equation

$$\begin{aligned}
 p(z) = & ((c_3 + c_2 - 2)c_1 + (c_3 - c_2)\delta)(\alpha + \beta) z^2 \\
 & + \left(((c_3 + c_2 - 2e^{-\delta})c_1 + (c_2 - c_3)\delta)\alpha \right. \\
 & \quad \left. + 2((1 - e^{-\delta})c_1 + (c_2 - c_3)\delta)\beta \right) z \\
 & - 2(z^2 - z(c_3 + c_2) + e^{-\delta})c_1\gamma z^{1+d} \\
 & + ((2e^{-\delta} - c_3 - c_2)c_1 + (c_3 - c_2)\delta)\beta
 \end{aligned}$$

with

$$\begin{aligned}
 c_1 &= \sqrt{\delta^2 - 4\gamma} \\
 c_2 &= e^{-(\delta+c_1)/2} \\
 c_3 &= e^{-(\delta-c_1)/2}
 \end{aligned}$$

► Mass m and Sampling Period T dropped out!



System Description

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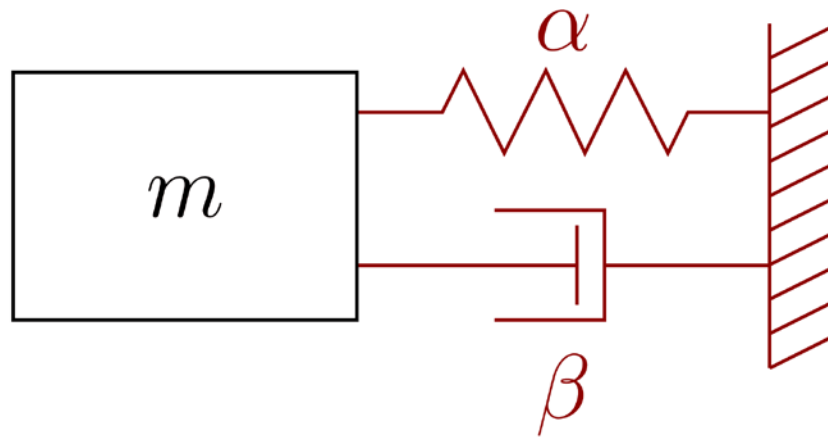
► Mass m and Sampling Period T dropped out!



Normalized Stability Boundaries

$$\begin{aligned}d &= t_d / T \\ \alpha &= KT^2 / m \\ \beta &= BT / m \\ \gamma &= kT^2 / m \\ \delta &= bT / m\end{aligned}$$

Simple case: $\gamma = \delta = 0$



$$d = t_d / T$$

$$\alpha = KT^2 / m$$

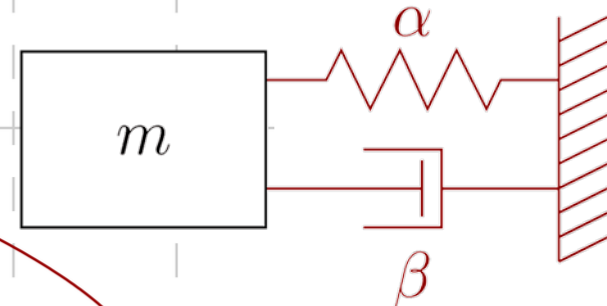
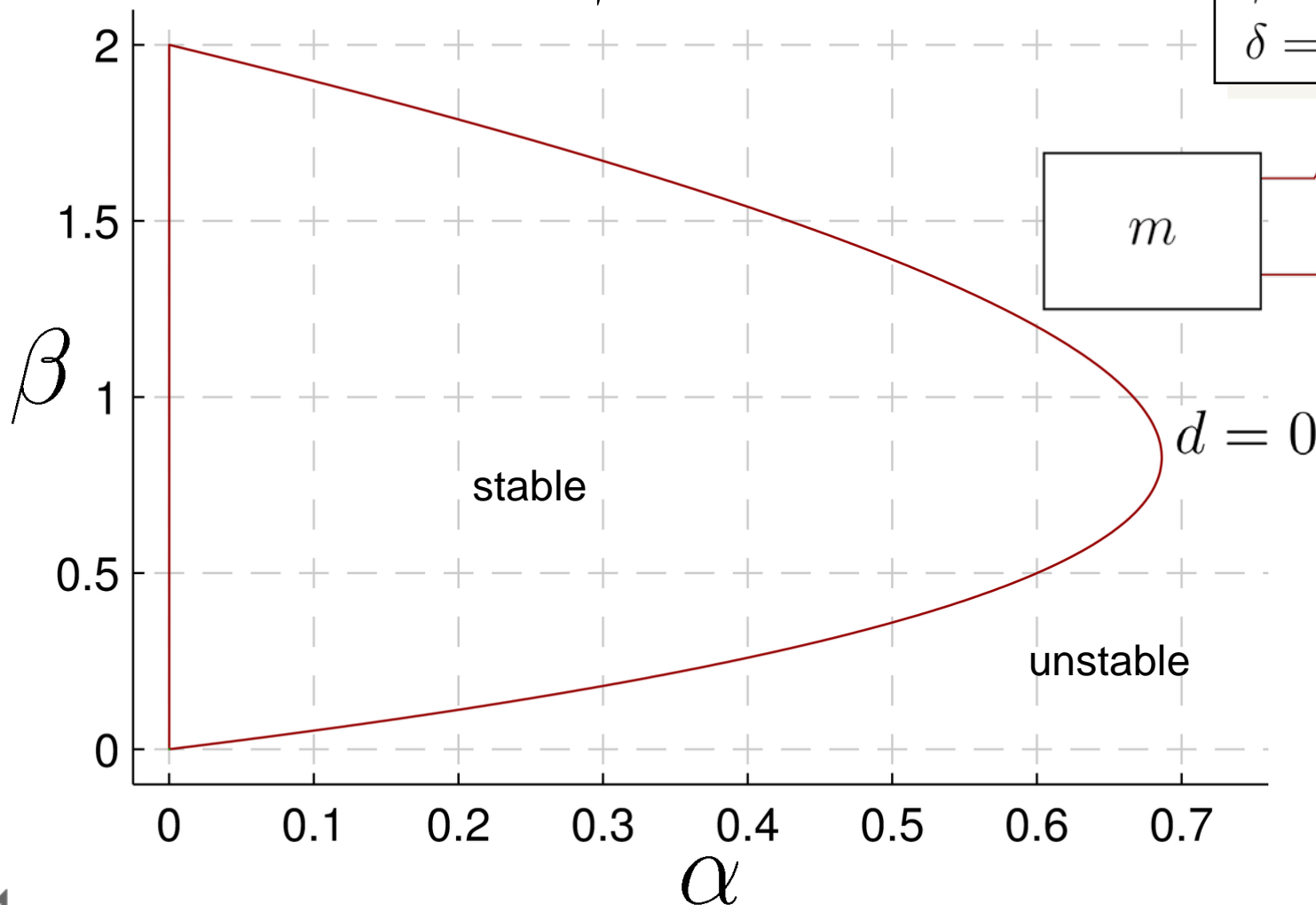
$$\beta = BT / m$$

$$\gamma = kT^2 / m$$

$$\delta = bT / m$$

Normalized Stability Boundaries

$$\gamma = \delta = 0$$



$$d = t_d / T$$

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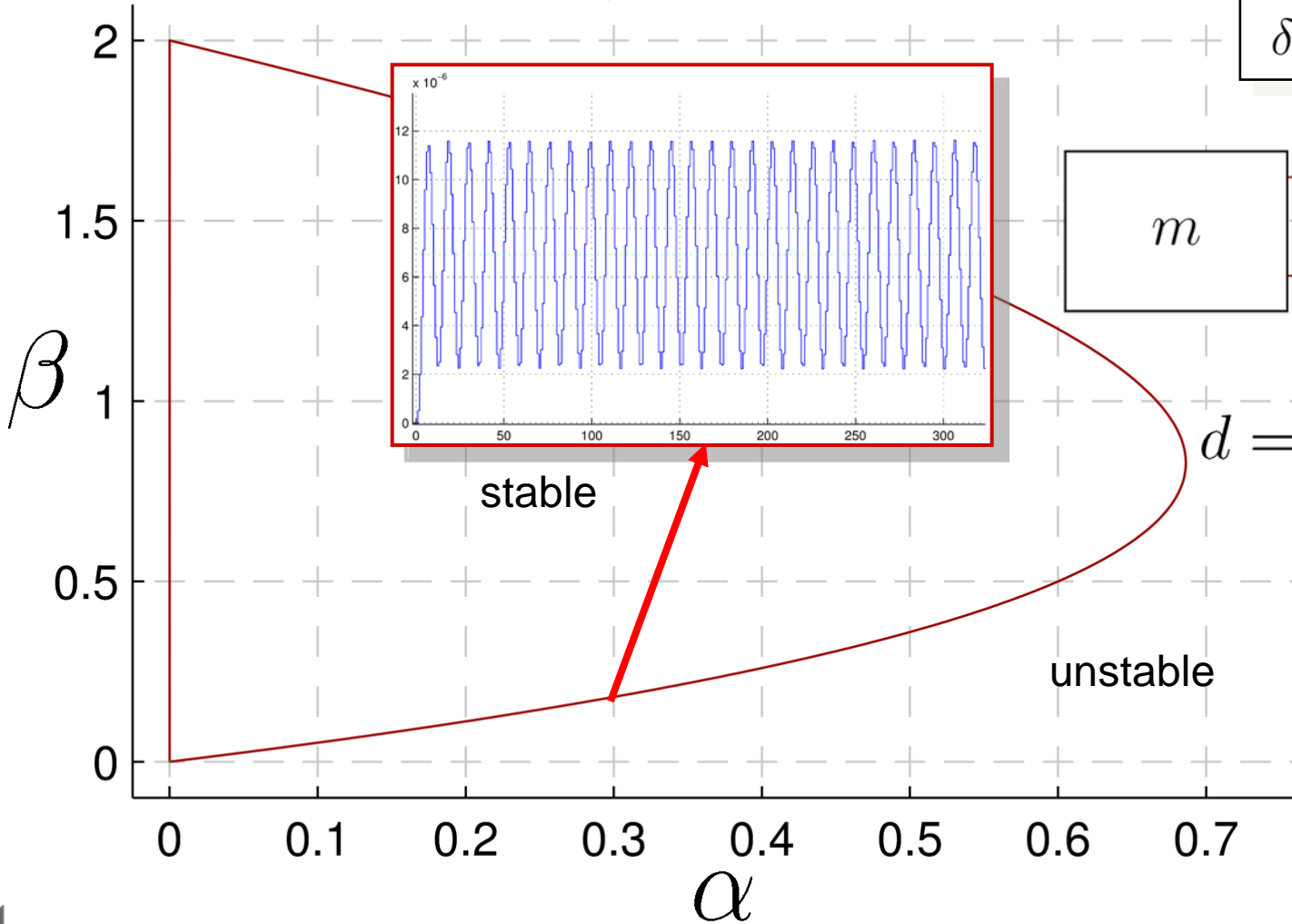
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Normalized Stability Boundaries

$$\gamma = \delta = 0$$



$d = 0$



$$d = t_d / T$$

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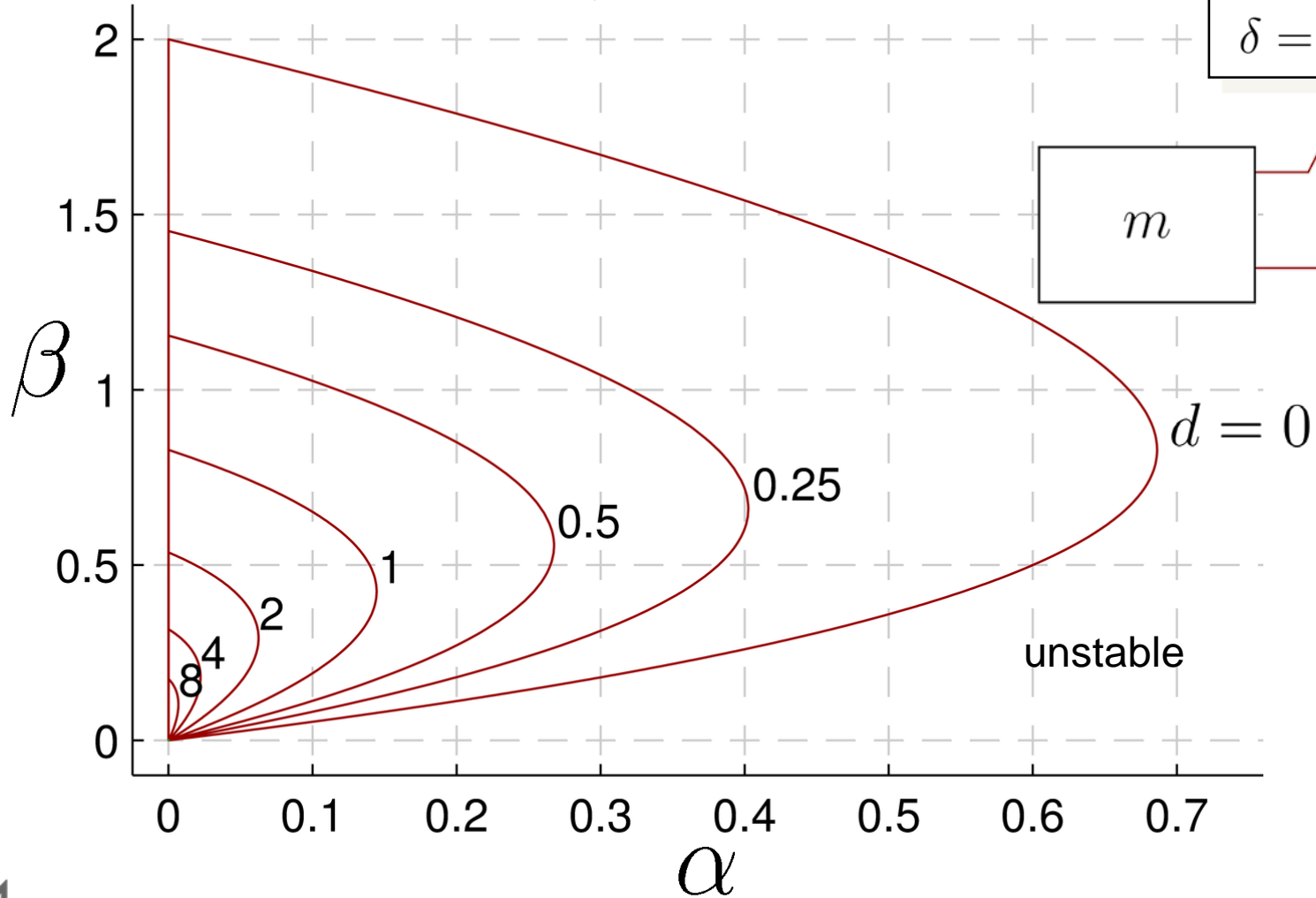
$$\beta = BT / m$$

$$\gamma = kT^2 / m$$

$$\delta = bT / m$$

Normalized Stability Boundaries

$$\gamma = \delta = 0$$



Realistic Parameter Range

for haptic devices holds

$$b_L/m_L < 0.625 s^{-1}$$

$$T \leq 0.001 s$$

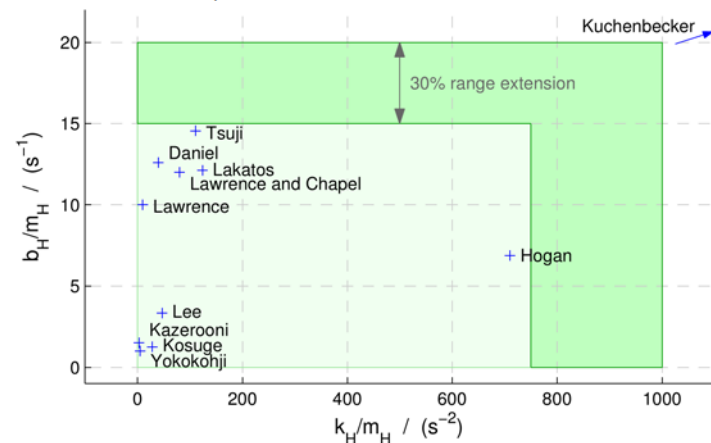
Device	m [Kg]	b [Ns/m]	T [ms]
Delta	0.250	0.01	0.33
Freedom 6	0.250	0.01	1
Impulse Engine	0.032	0.02	0.2
MIT Toolhandle	0.119	0.001	1
Omega	0.220	0.01	0.33
Phantom 1.0	0.072	0.005	1

[Diolaiti 2006]

for human arms holds

$$k_H/m_H \leq 710 s^{-2}$$

$$b_H/m_H \leq 14.6 s^{-1}$$



[Gil 2004, Hulin 2014]

+ 30%

$$0 \leq \gamma = k \cdot T^2 / m \leq 1 \cdot 10^{-3}$$

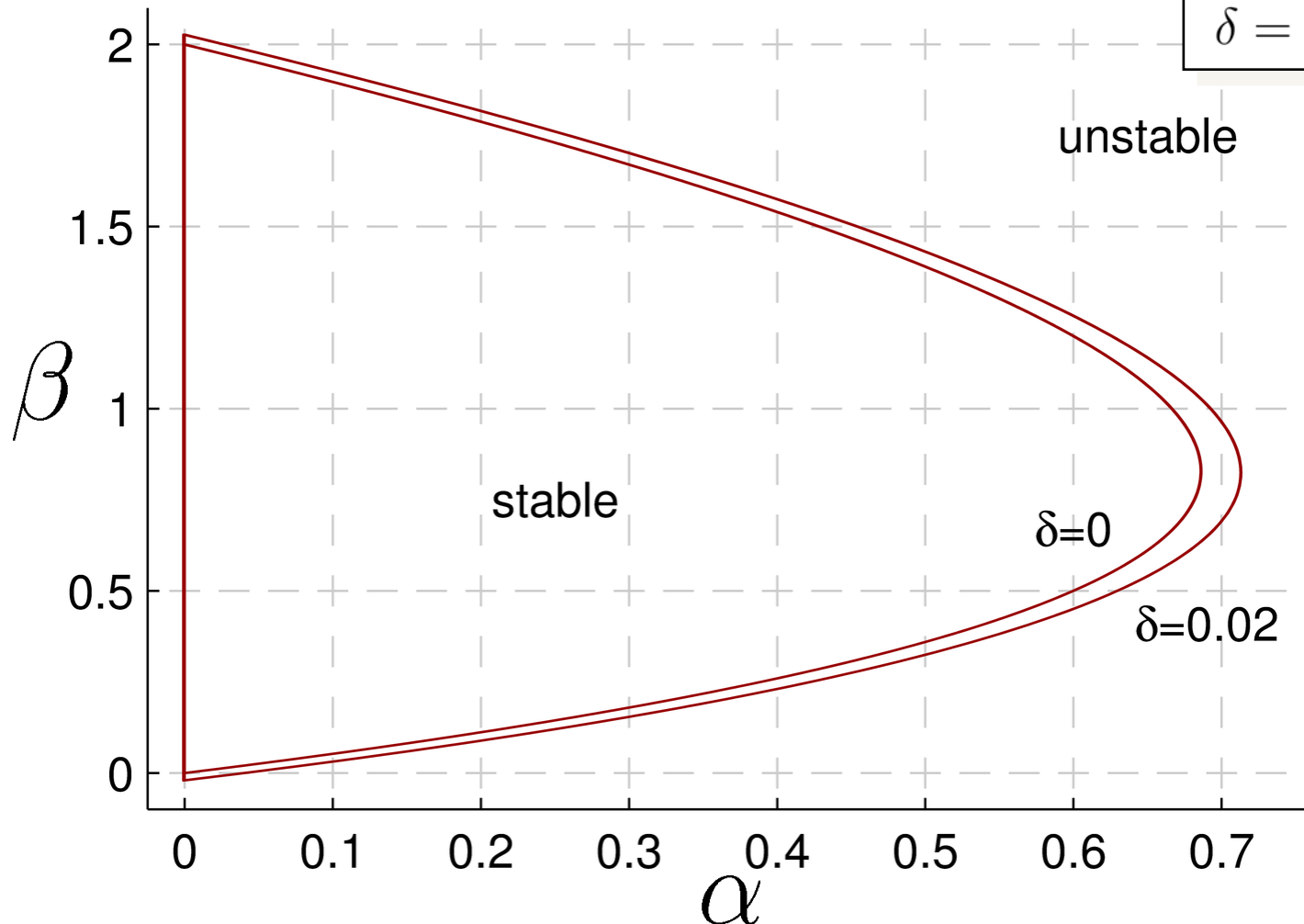
$$0 \leq \delta = b \cdot T / m \leq 20 \cdot 10^{-3}$$



$$\begin{aligned}d &= t_d / T \\ \alpha &= KT^2 / m \\ \beta &= BT / m \\ \gamma &= kT^2 / m \\ \delta &= bT / m\end{aligned}$$

Normalized Stability Boundaries

$$\gamma \in \{0, 0.001\}, \delta \in \{0, 0.02\}, d=0$$



$$d = t_d / T$$

$$\alpha = KT^2 / m$$

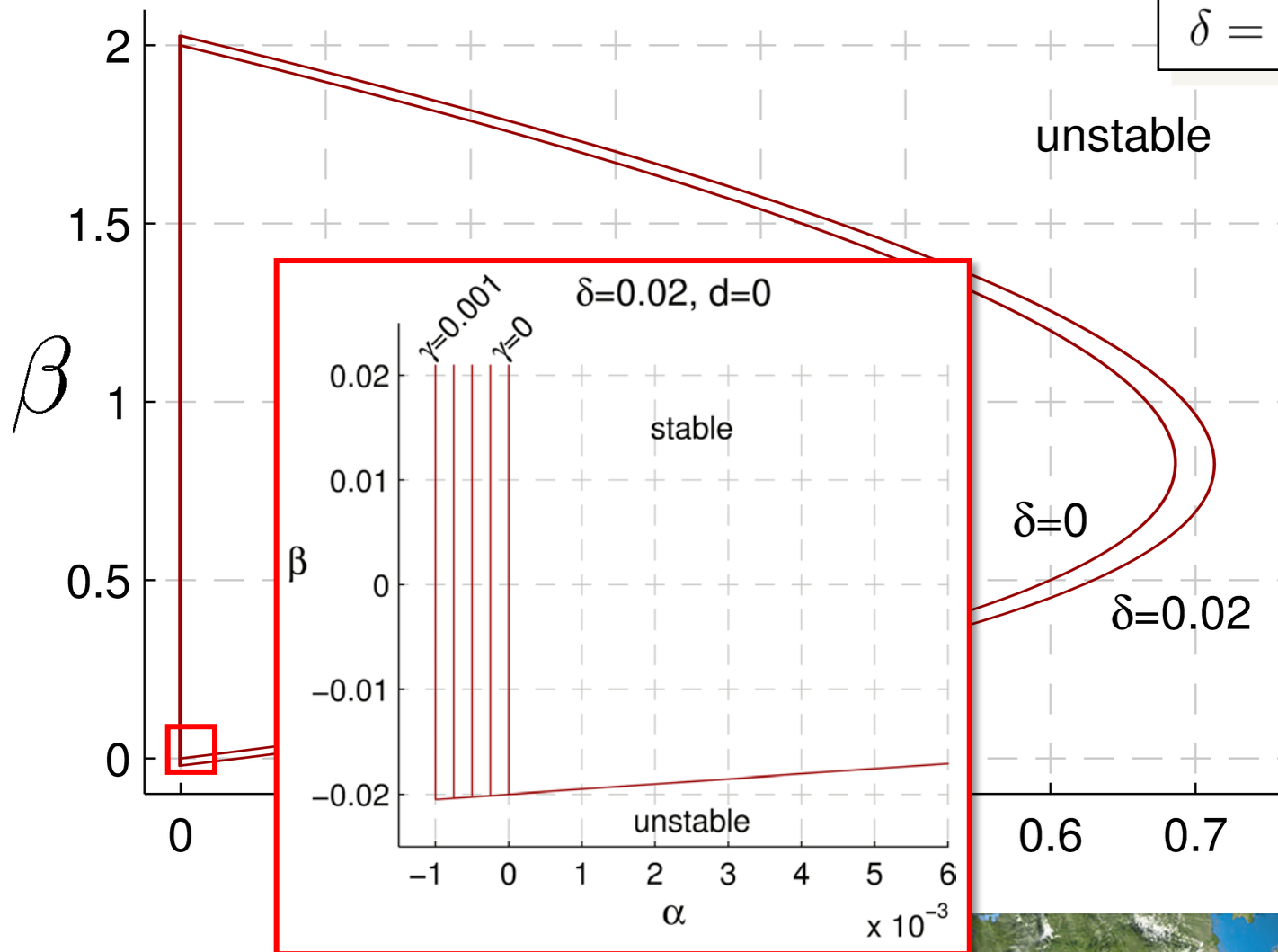
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$$d = t_d / T$$

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$\gamma \in \{0, 0.001\}$, $\delta \in \{0, 0.02\}$, $d=0$

Sum of stiffnesses

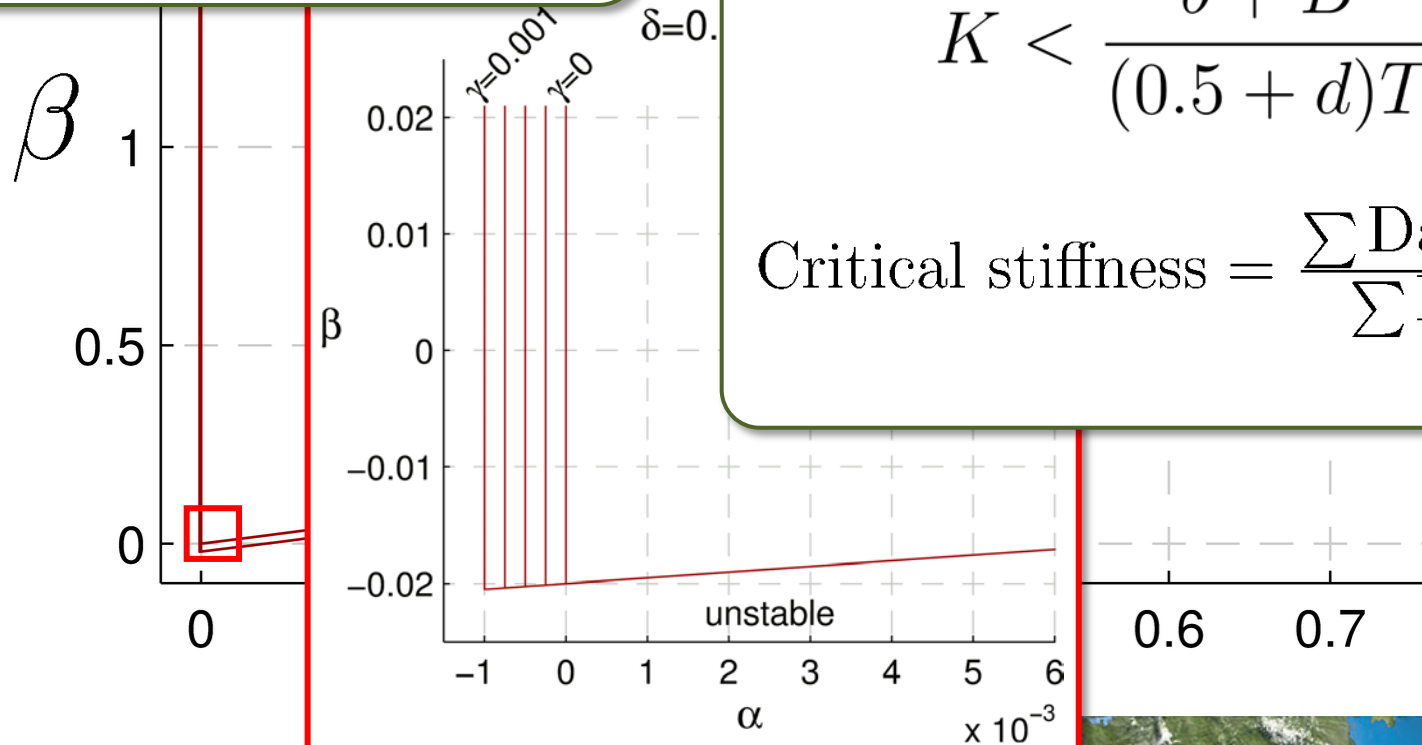
$$K + k > 0$$

Linear stability condition

[Gil & Hulin 2007]

$$K < \frac{b + B}{(0.5 + d)T}$$

$$\text{Critical stiffness} = \frac{\sum \text{Damping}}{\sum \text{Delay}}$$



$$d = t_d / T$$

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Normalized Stability Boundaries

$\gamma \in \{0, 0.001\}$, $\delta \in \{0, 0.02\}$, $d=0$

Sum of stiffnesses

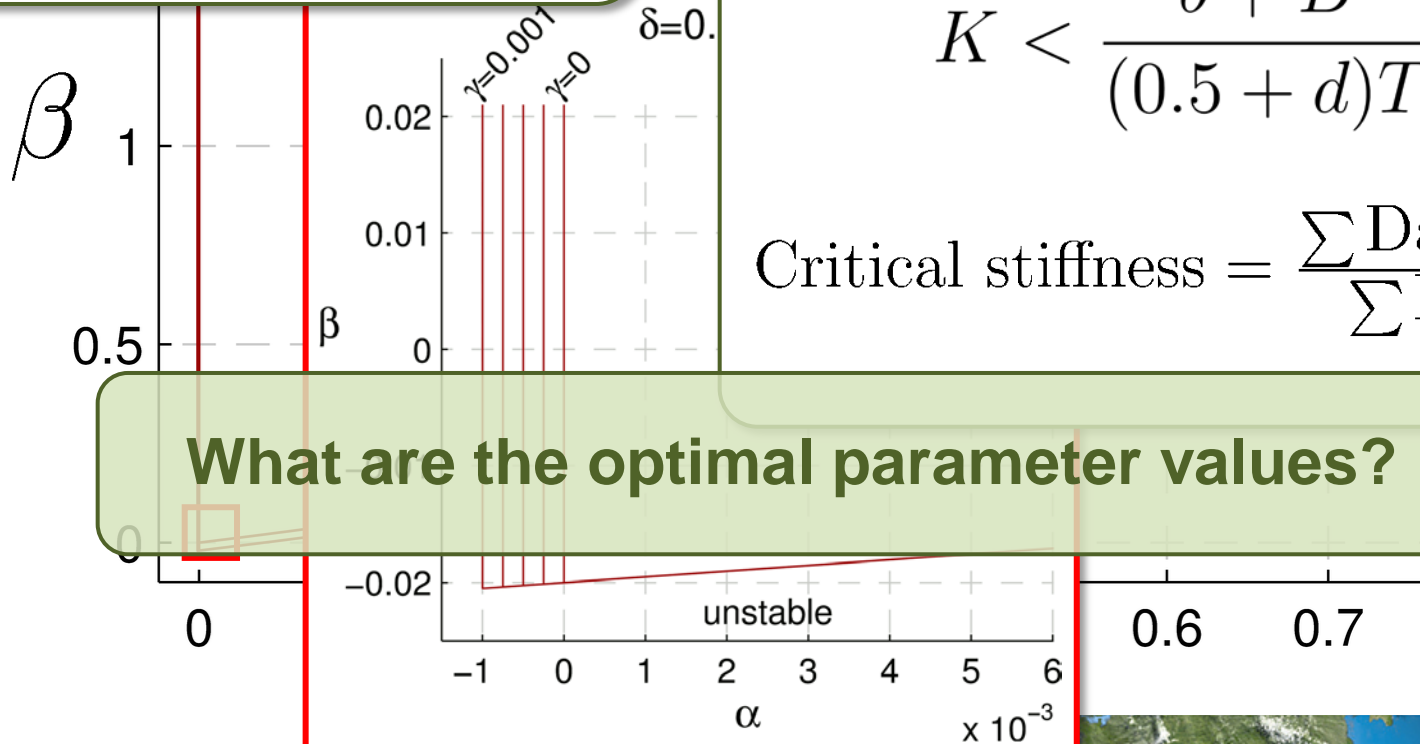
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Linear stability condition

[Gil & Hulin 2007]

$$K < \frac{b + B}{(0.5 + d)T}$$

$$\text{Critical stiffness} = \frac{\sum \text{Damping}}{\sum \text{Delay}}$$



What are the optimal parameter values?

Part 2: Control Design

Optimal Control for Haptic Interaction

[Hulin et al., SYROCO2006],
[Hulin et al., IROS2013],
[Hulin, RA-L/ICRA 2017]



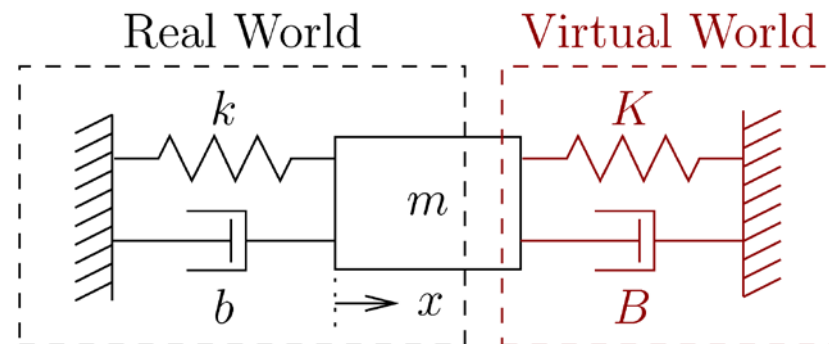
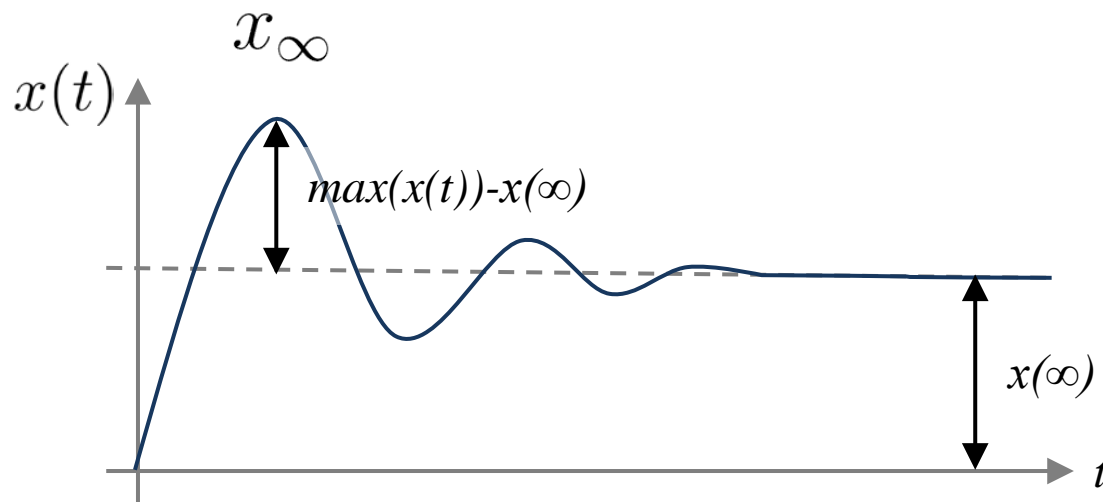
Response-based Control Design

Minimum Overshoot

Motivation: Optimal controller for the mass spring damper system: minimum overshoot

Cost function:

$$C_{\text{ov}} := \frac{\max(x(t)) - x_{\infty}}{x_{\infty}}$$



$$d = t_d / T$$

$$\alpha = KT^2 / m$$

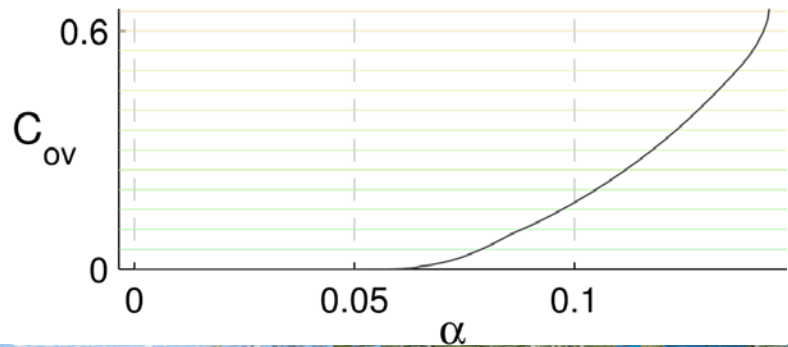
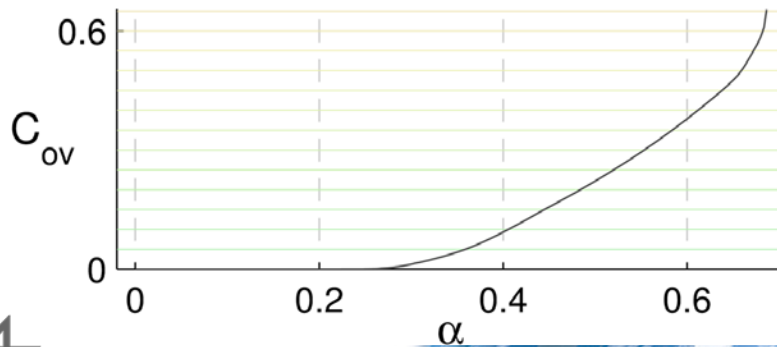
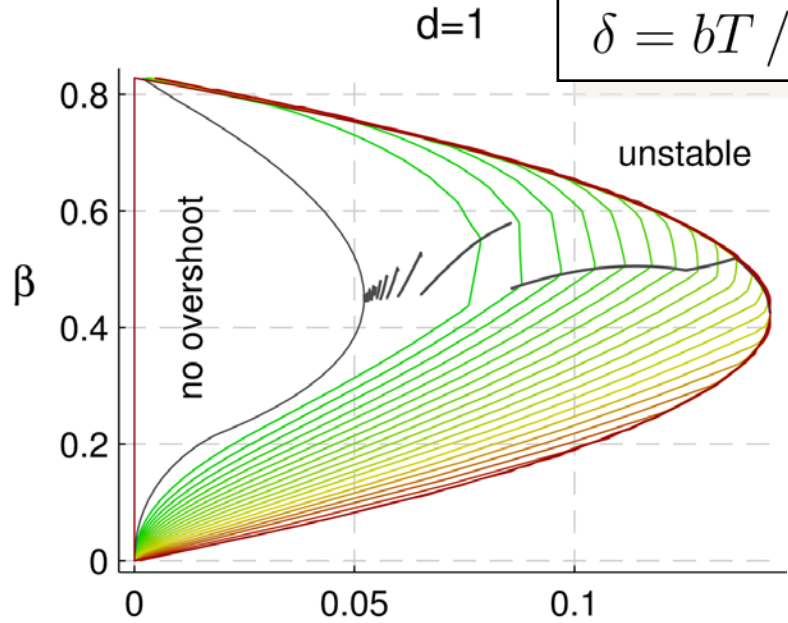
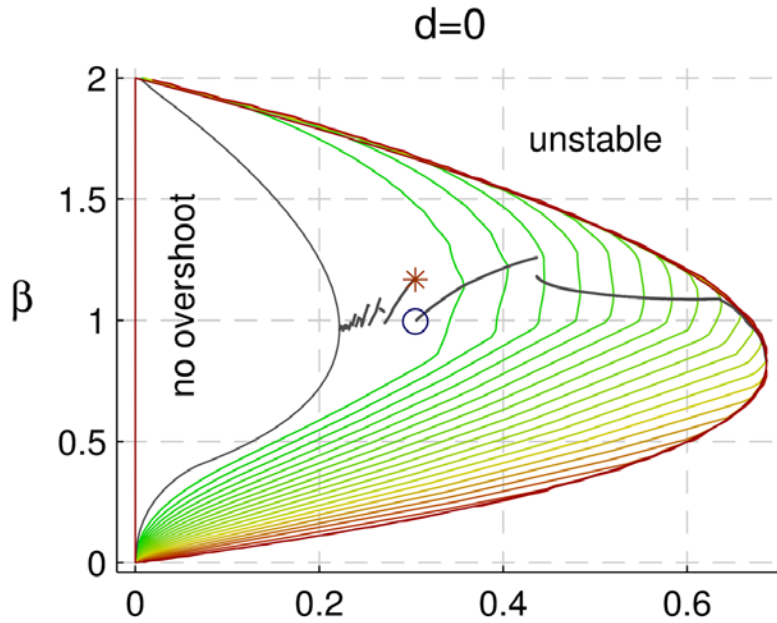
$$\beta = BT / m$$

$$\gamma = kT^2 / m$$

$$\delta = bT / m$$

Response-based Control Design

Minimum Overshoot



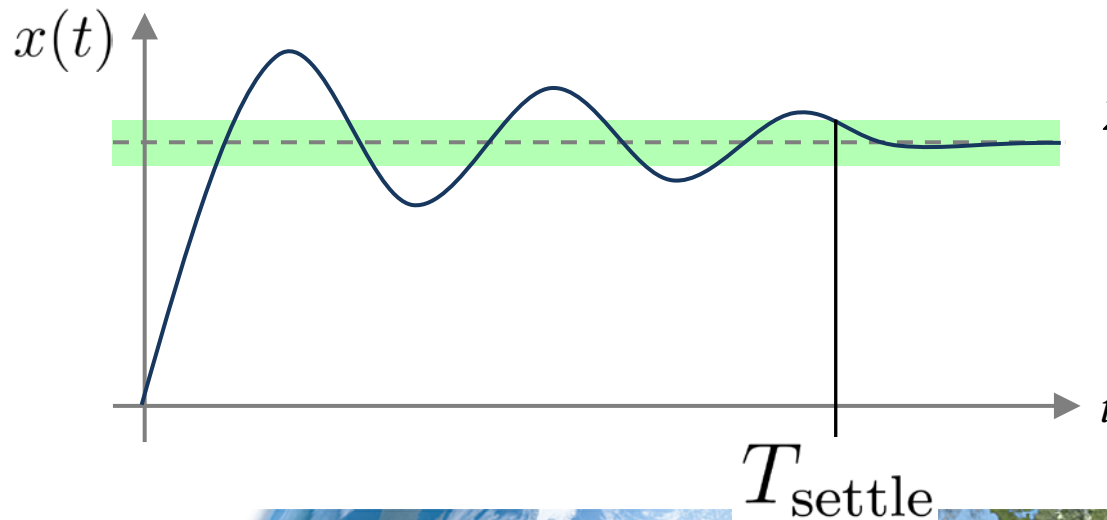
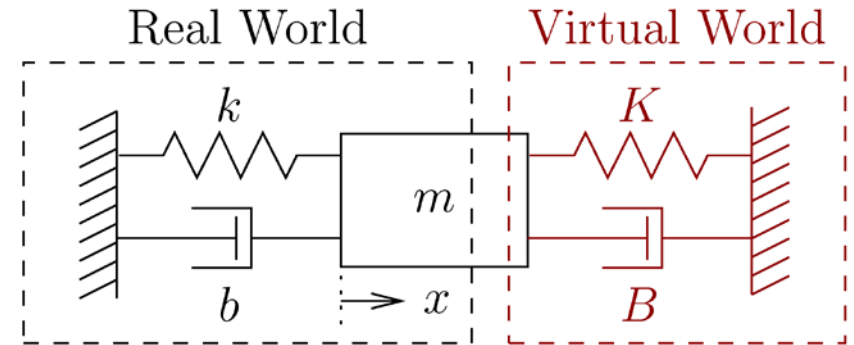
Response-based Control Design

Minimum Settling Time

Motivation: Optimal controller for the mass spring damper system: minimum settling time

Cost function:

$$C_{\text{settle}} := T_{\text{settle}}/T$$



2% threshold band
around $|x(t) - x(\infty)|$



$$d = t_d / T$$

$$\alpha = KT^2 / m$$

$$\beta = BT / m$$

$$\gamma = kT^2 / m$$

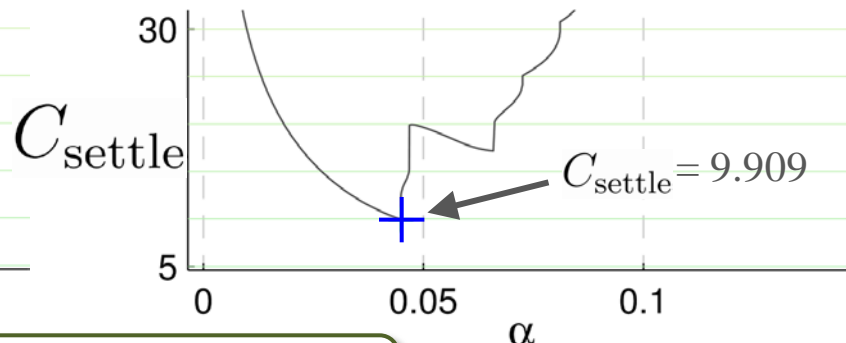
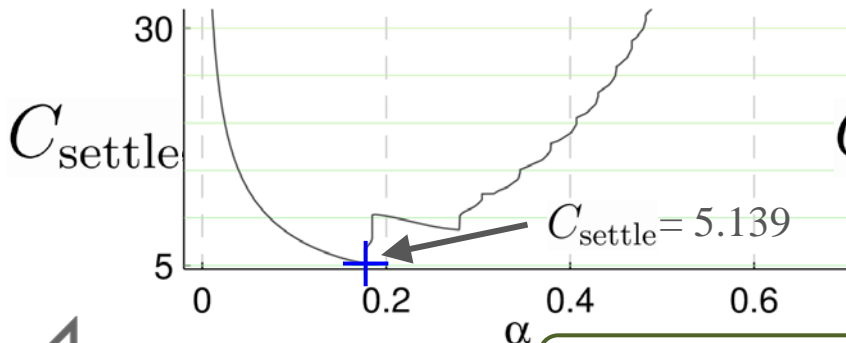
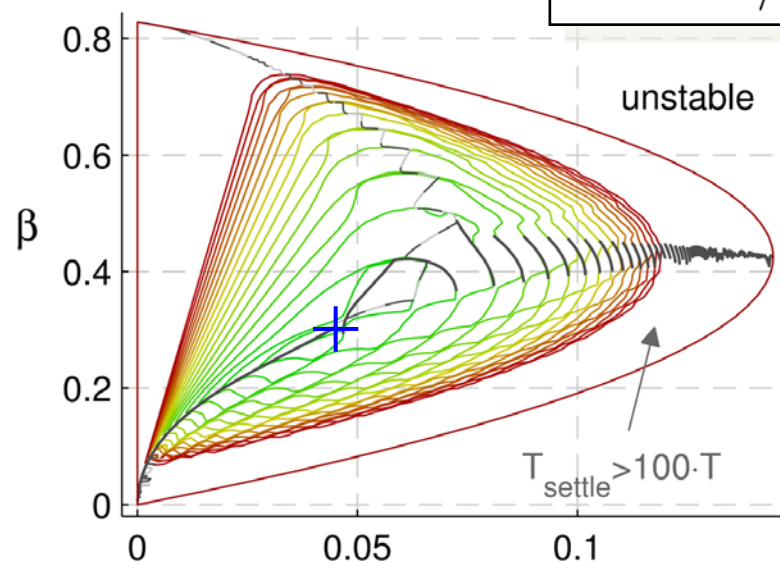
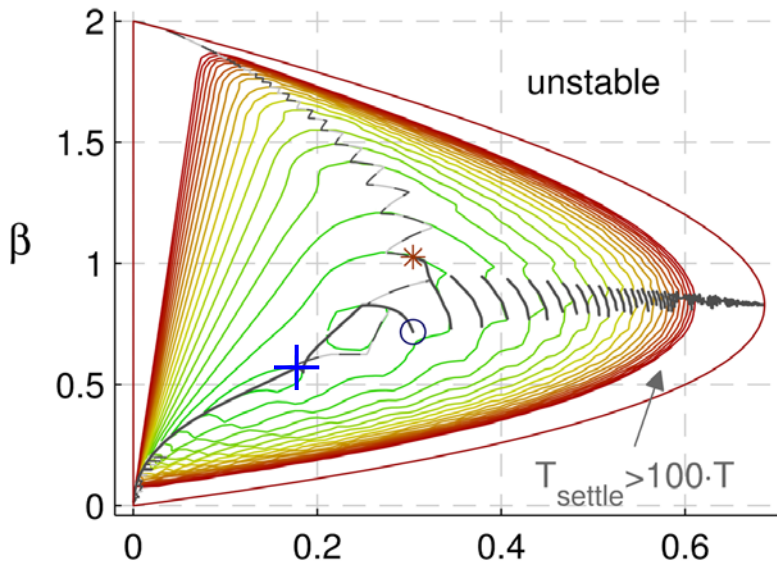
$$\delta = bT / m$$

Response-based Control Design

Minimum Settling Time

d=0

d=1



$$C_{\text{settle}} := T_{\text{settle}} / T$$



$$d = t_d / T$$

$$\alpha = KT^2 / m$$

$$\beta = BT / m$$

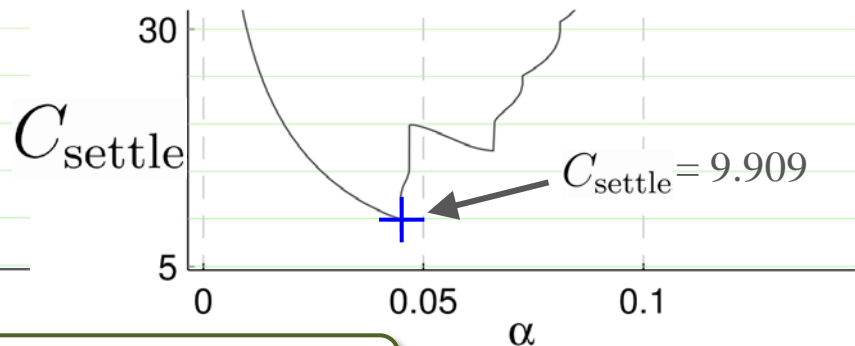
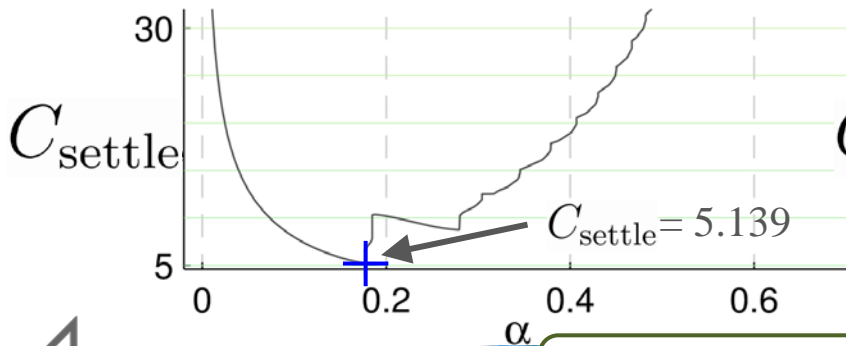
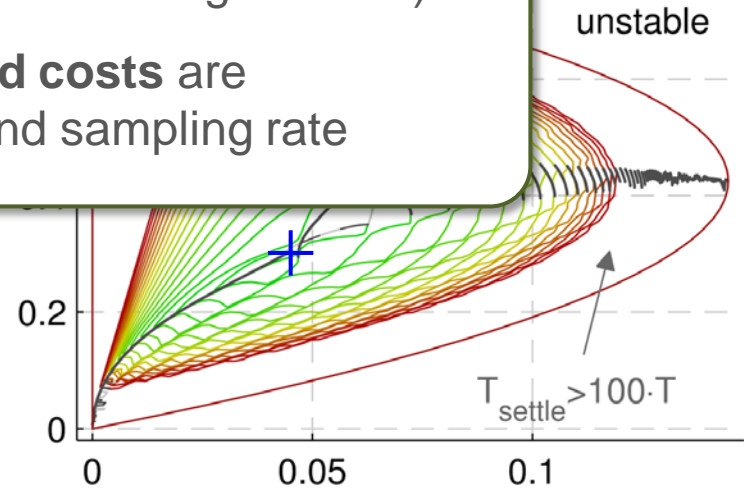
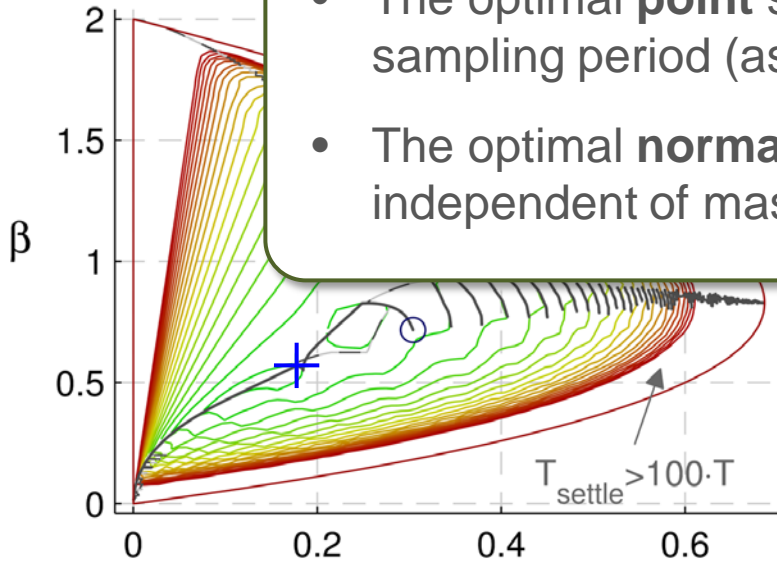
$$\gamma = kT^2 / m$$

$$\delta = bT / m$$

Response-based Control Design

Minimum Settling Time

- The optimal **point** scales with mass and sampling period (as the stable region does)
- The optimal **normalized costs** are independent of mass and sampling rate



$$C_{\text{settle}} := T_{\text{settle}} / T$$



$$d = t_d / T$$

$$\alpha = KT^2 / m$$

$$\beta = BT / m$$

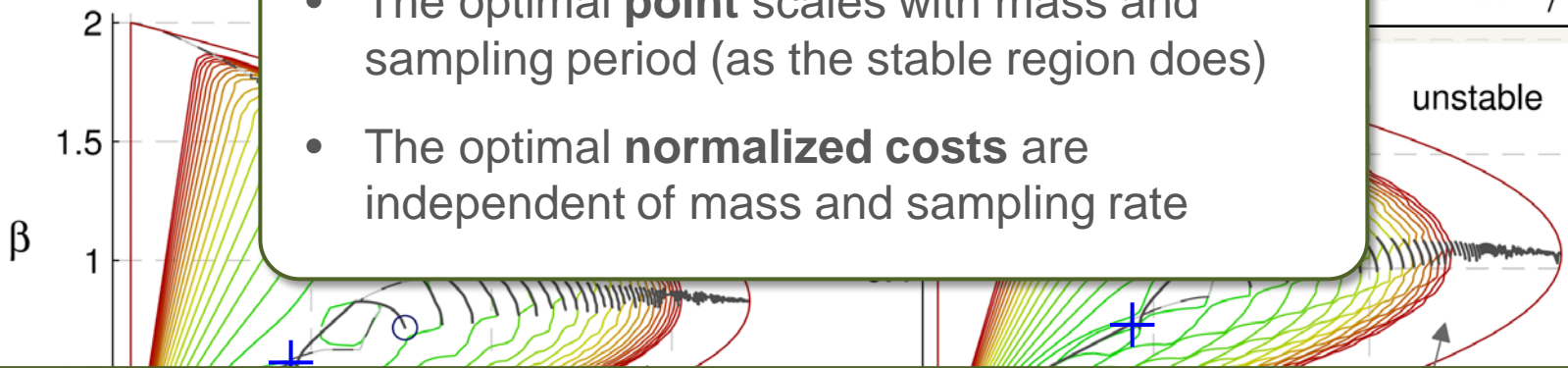
$$\gamma = kT^2 / m$$

$$\delta = bT / m$$

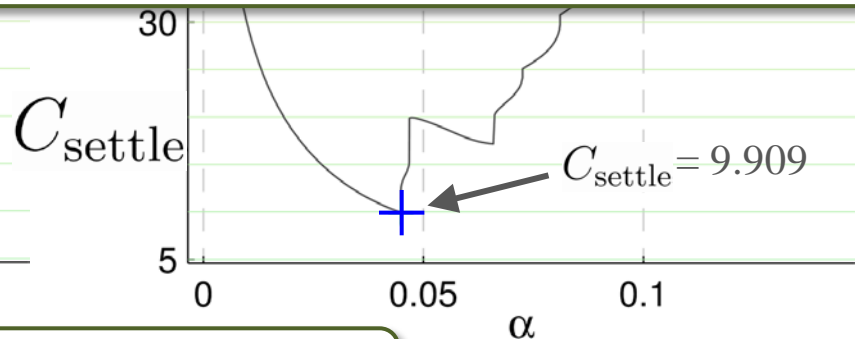
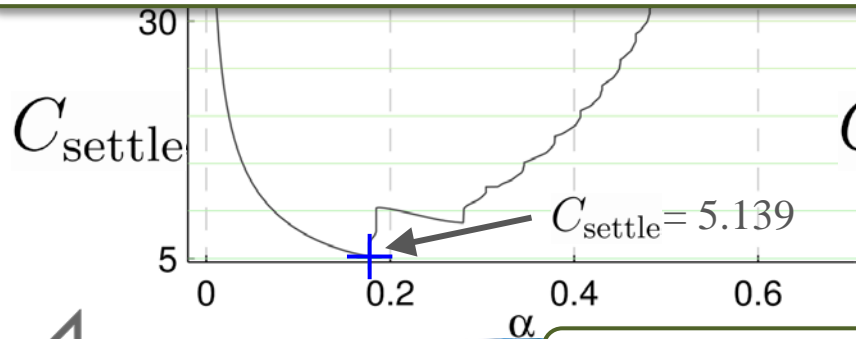
Response-based Control Design

Minimum Settling Time

- The optimal **point** scales with mass and sampling period (as the stable region does)
- The optimal **normalized costs** are independent of mass and sampling rate



How do the costs evolve for higher time delays?

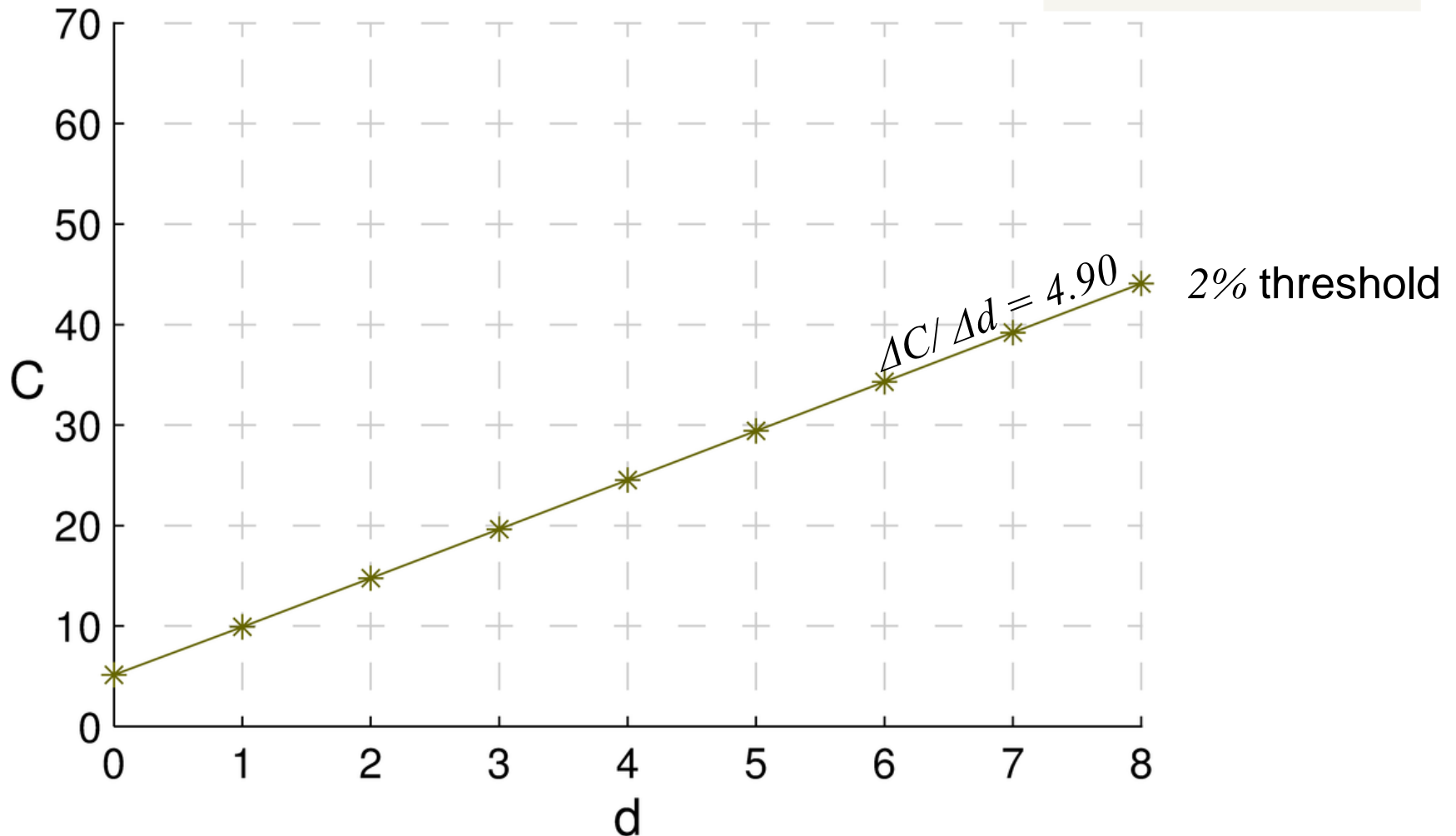


$$C_{\text{settle}} := T_{\text{settle}} / T$$

Control Design

Influence of Delay

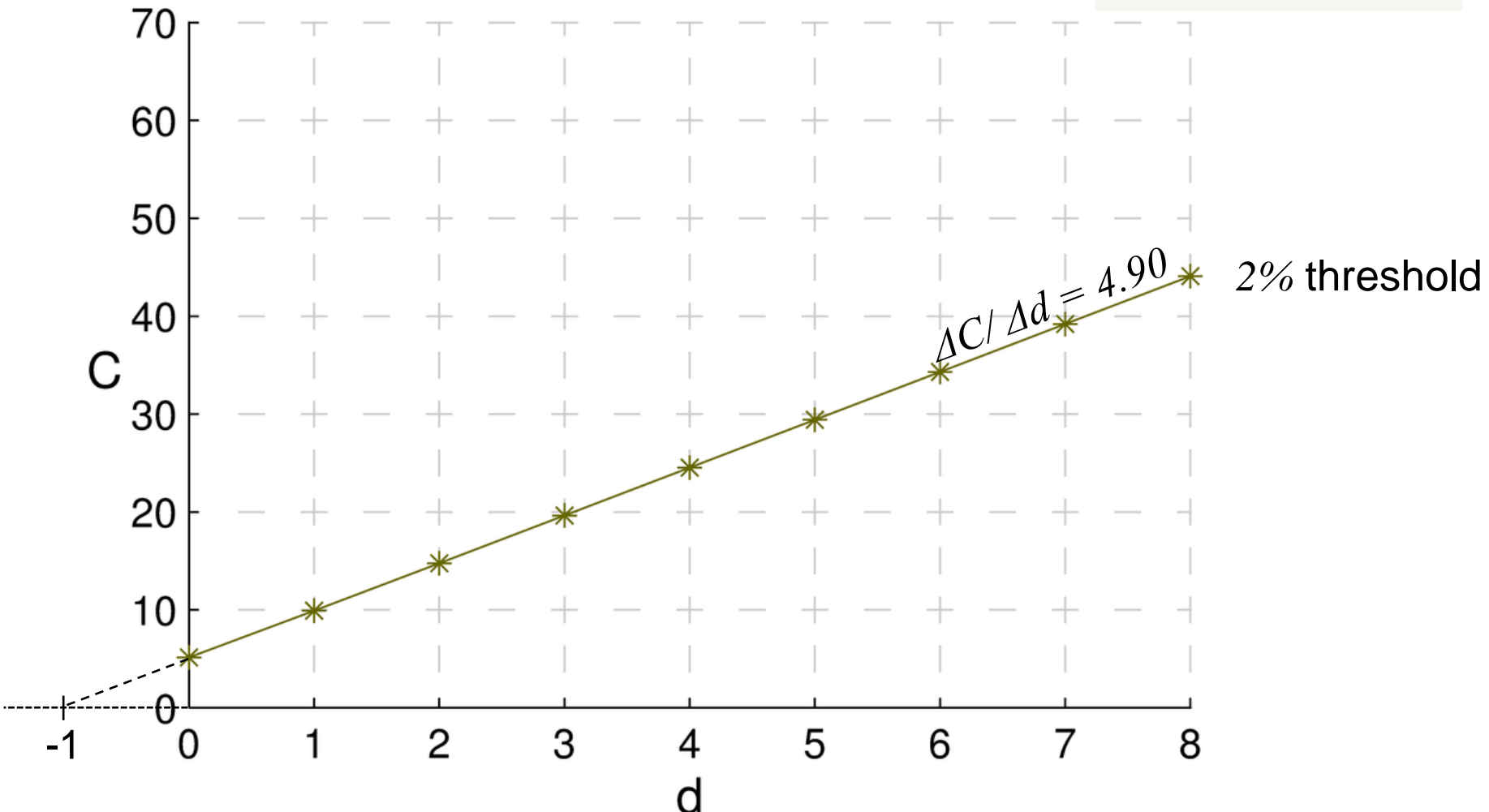
$$t_d = d \cdot T$$



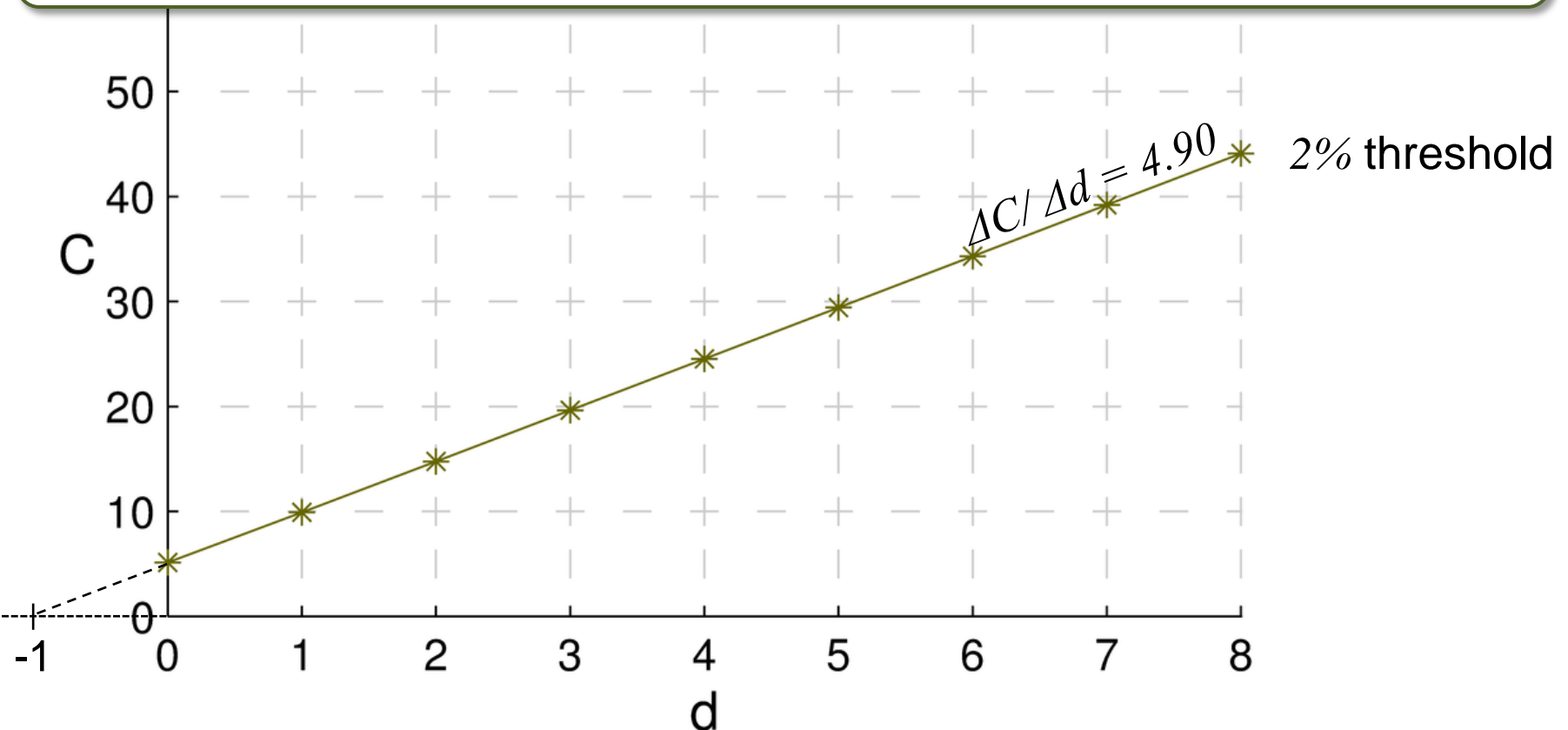
Control Design

Influence of Delay

$$t_d = d \cdot T$$



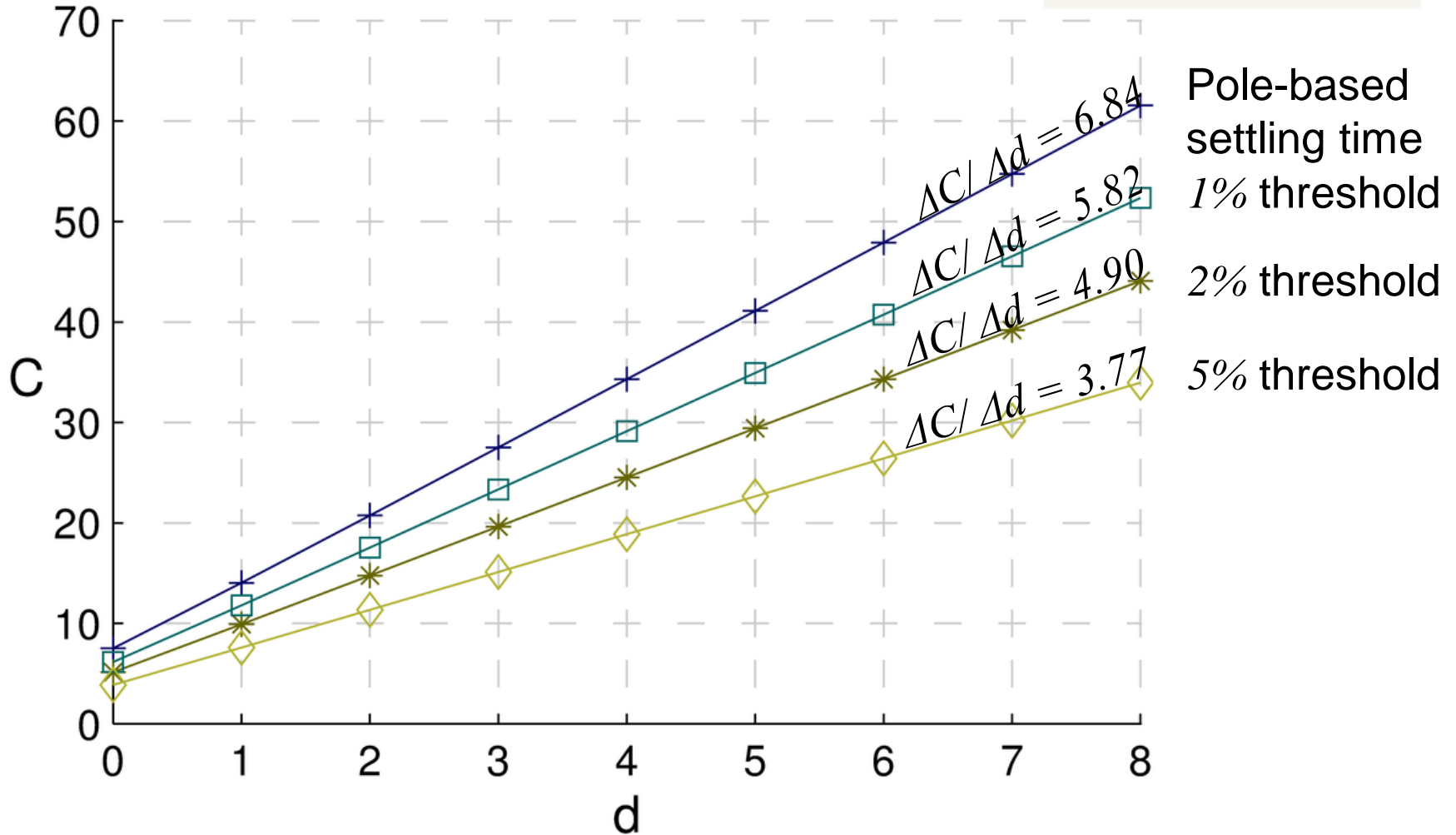
1. Each sampling period of additional time delay causes the optimal settling time to increase by approximately five sampling periods.
2. The effect of discrete-time sampling corresponds to a delay of one whole sampling period in terms of cost.



Control Design

Influence of Delay

Time Delay
 $t_d = d \cdot T$



Part 3: Experiments

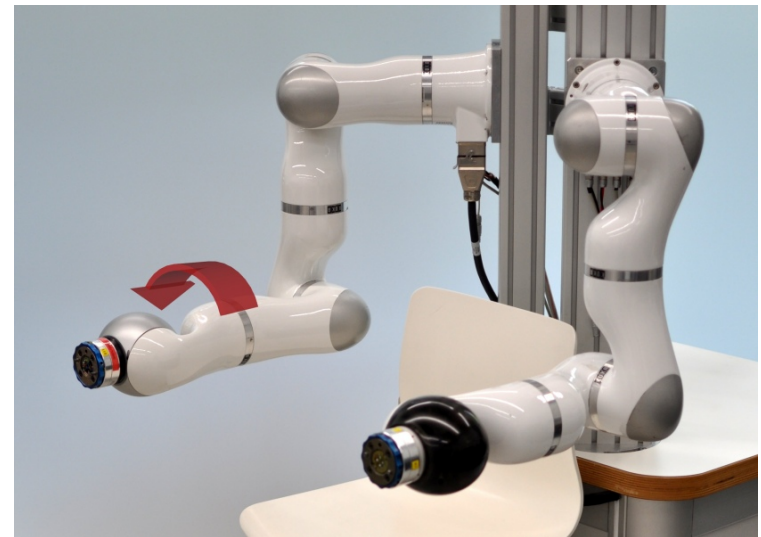
Verification of the theoretical approach on real systems

[Hulin et al., IROS2013],
[Hulin, RA-L/ICRA 2017]



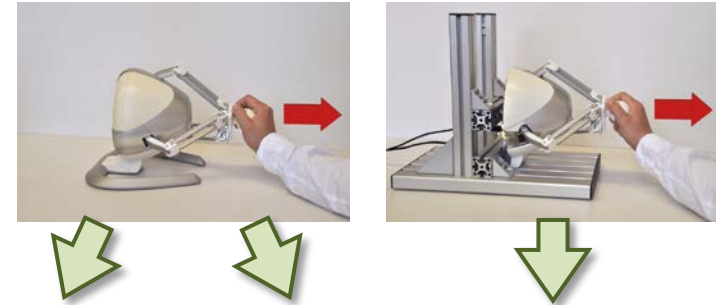
Experiments

- Two different devices with widely differing properties:
 - Novint Falcon
 - DLR/KUKA Light-Weight Robot (the right arm of HUG)
- Influence of human operator (Falcon)
- Influence of time delay (LWR)
- Influence of modifications in the HW (Falcon)



Experiments on the Novint Falcon

Experimental Procedure



A set of seven experiments:

- Influence of human operator
- Influence of modifications in the HW

human grip	Falcon A	Falcon B	Falcon B stiff
no operator	1	2	3
comfortable grip	4	—	5
firm grip	6	—	7

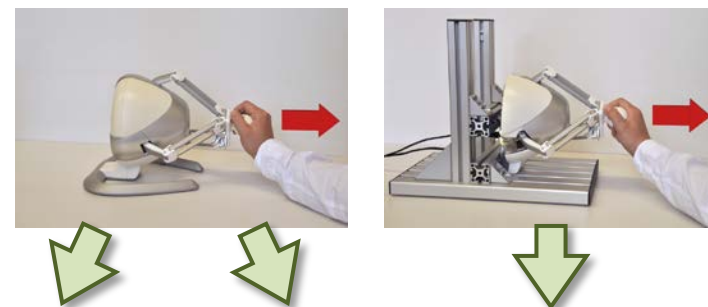


Experiments on the Novint Falcon

Experimental Procedure

A set of seven experiments:

- Influence of human operator
- Influence of modifications in the HW



human grip	Falcon A	Falcon B	Falcon B stiff
no operator	1	2	3
comfortable grip	4	—	5
firm grip	6	—	7

For each experiment a grid was defined:

no human

$$\Delta K=200\text{N/m}$$

$$\Delta B=4\text{Ns/m}$$



~2200 grid points

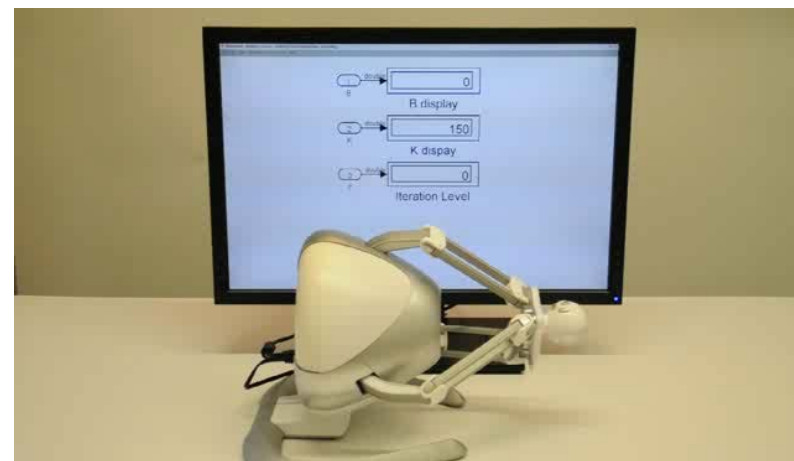
with human

$$\Delta K=400\text{N/m},$$

$$\Delta B=10\text{Ns/m}$$



~650 grid points



Experiments on the Novint Falcon

Parameter Estimation



Determine physical parameters to draw theoretical curves:

- Mass: conservation of linear momentum

$$m = \int_{t_1}^{t_2} F(\tau) d\tau / (\dot{x}(t_2) - \dot{x}(t_1))$$

- Viscous damping: negative virtual damping

human grip	dynamic mass m	viscous damping b	
	Falcon A&B ^a	Falcon A	Falcon B
no operator	0.58 kg	4 Ns/m	2 Ns/m
comfortable grip	0.65 kg	9 Ns/m	7 Ns/m
firm grip	1.00 kg	34 Ns/m	32 Ns/m

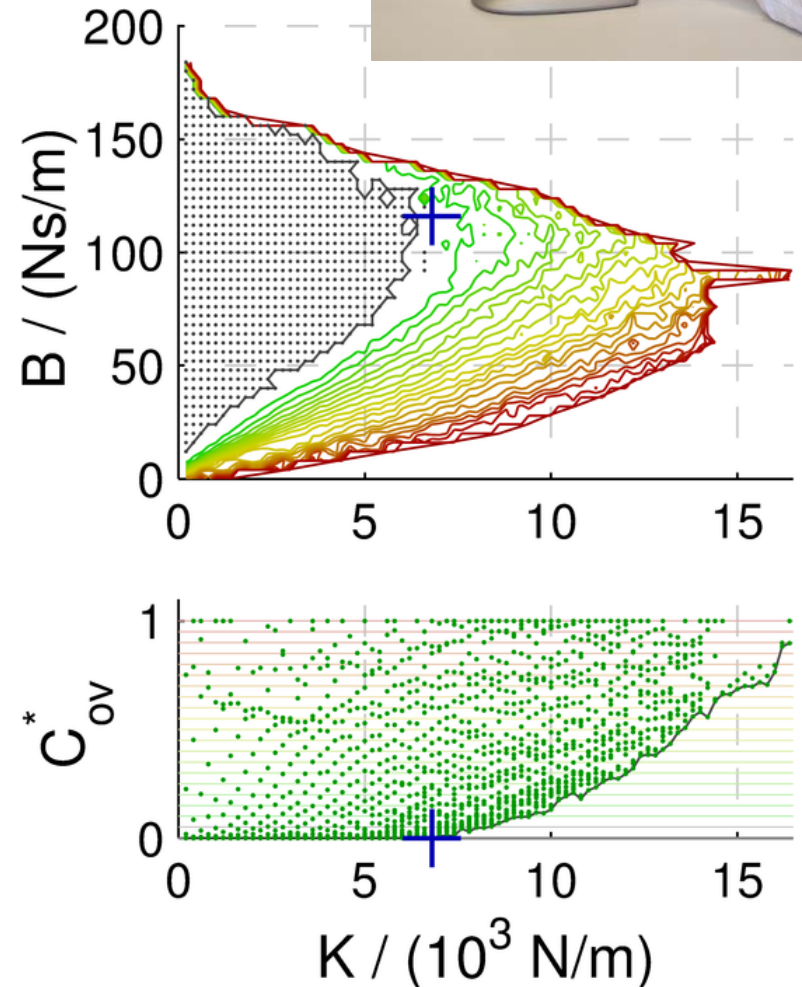
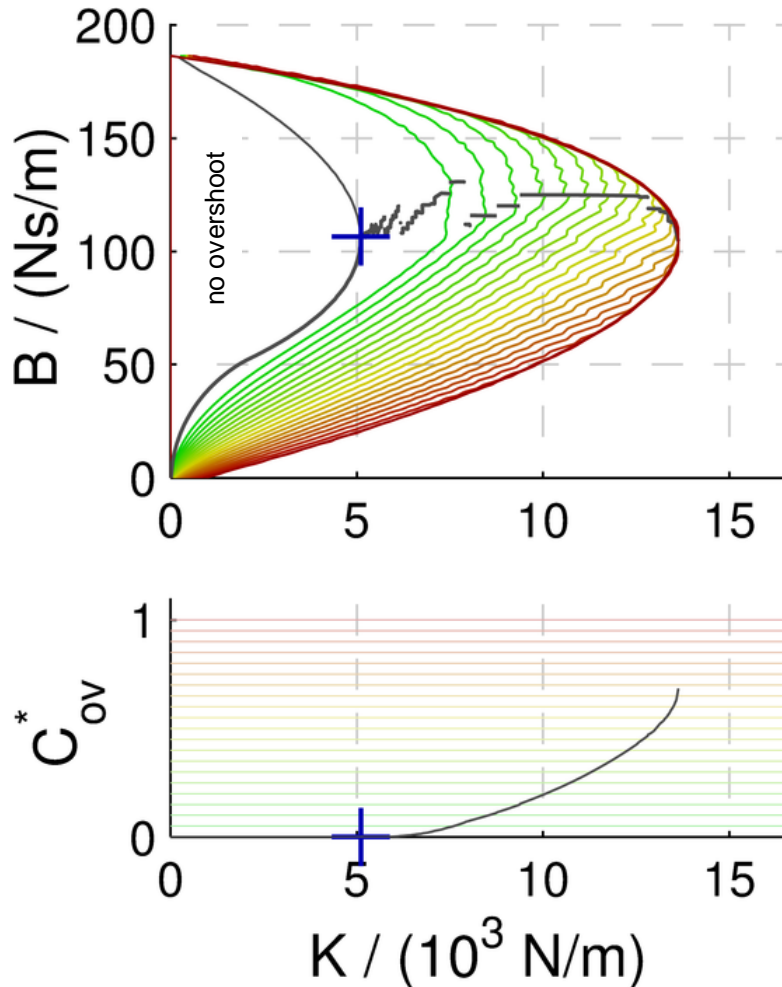
- Delay: first change in step response

$$3 \text{ ms} \leq t_d < 6 \text{ ms} \quad (\text{average delay: } t_d = 5 \text{ ms})$$



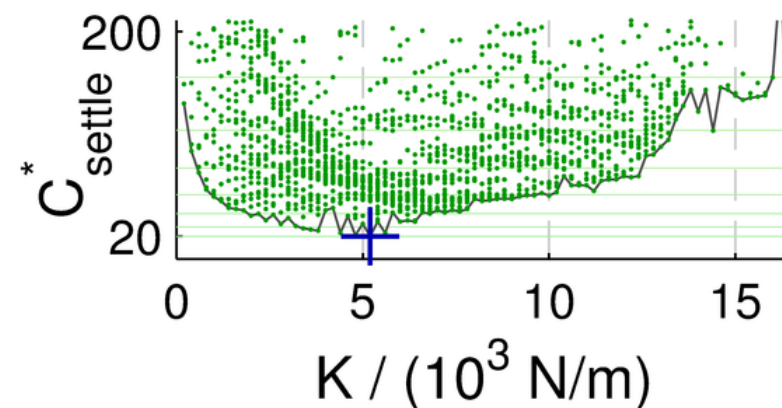
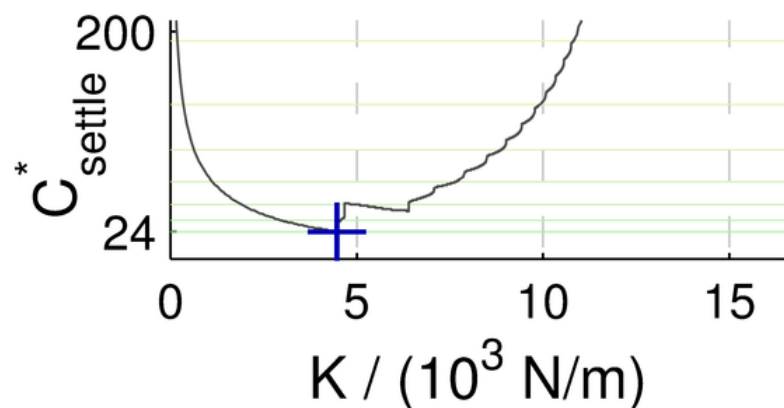
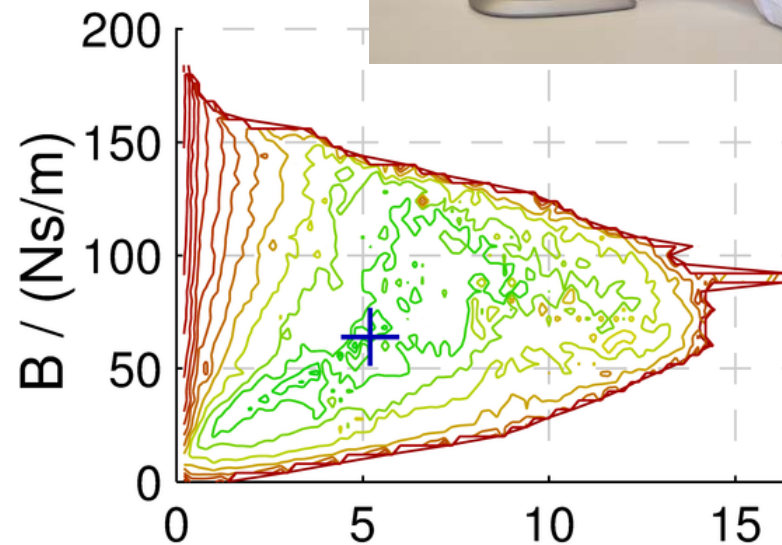
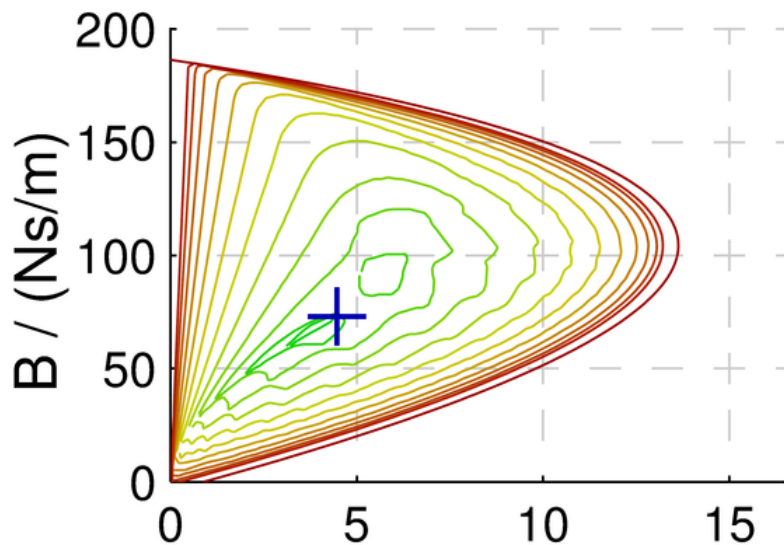
Experiments on the Novint Falcon

Results: Overshoot



Experiments on the Novint Falcon

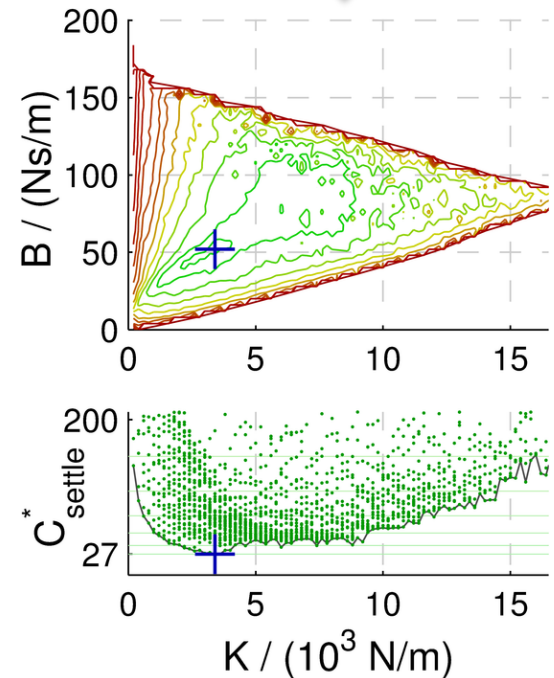
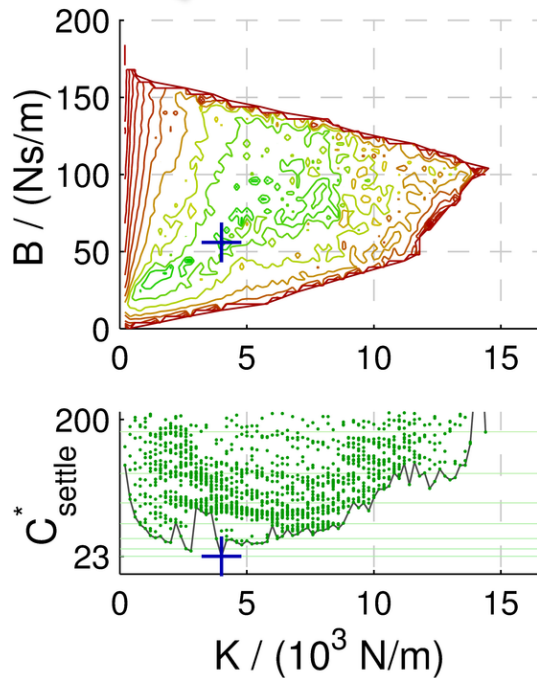
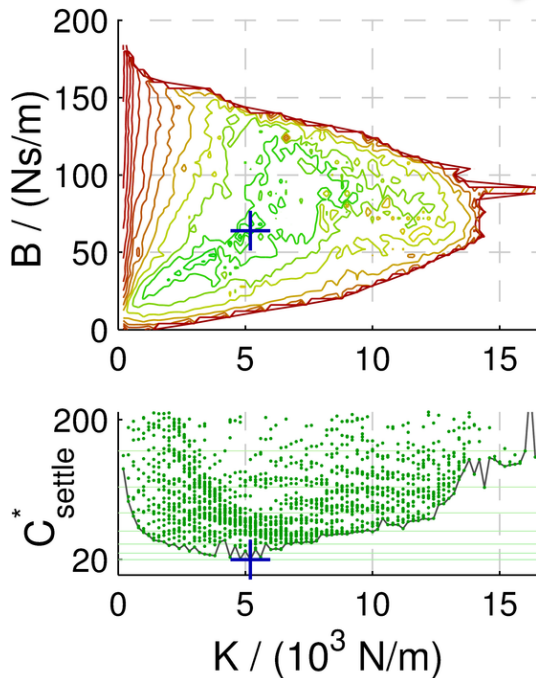
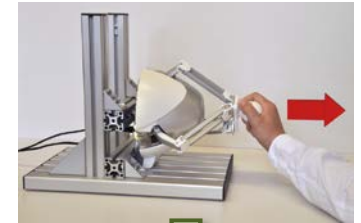
Results: 2% Settling Time



Experiments on the Novint Falcon

Results: 2% Settling Time

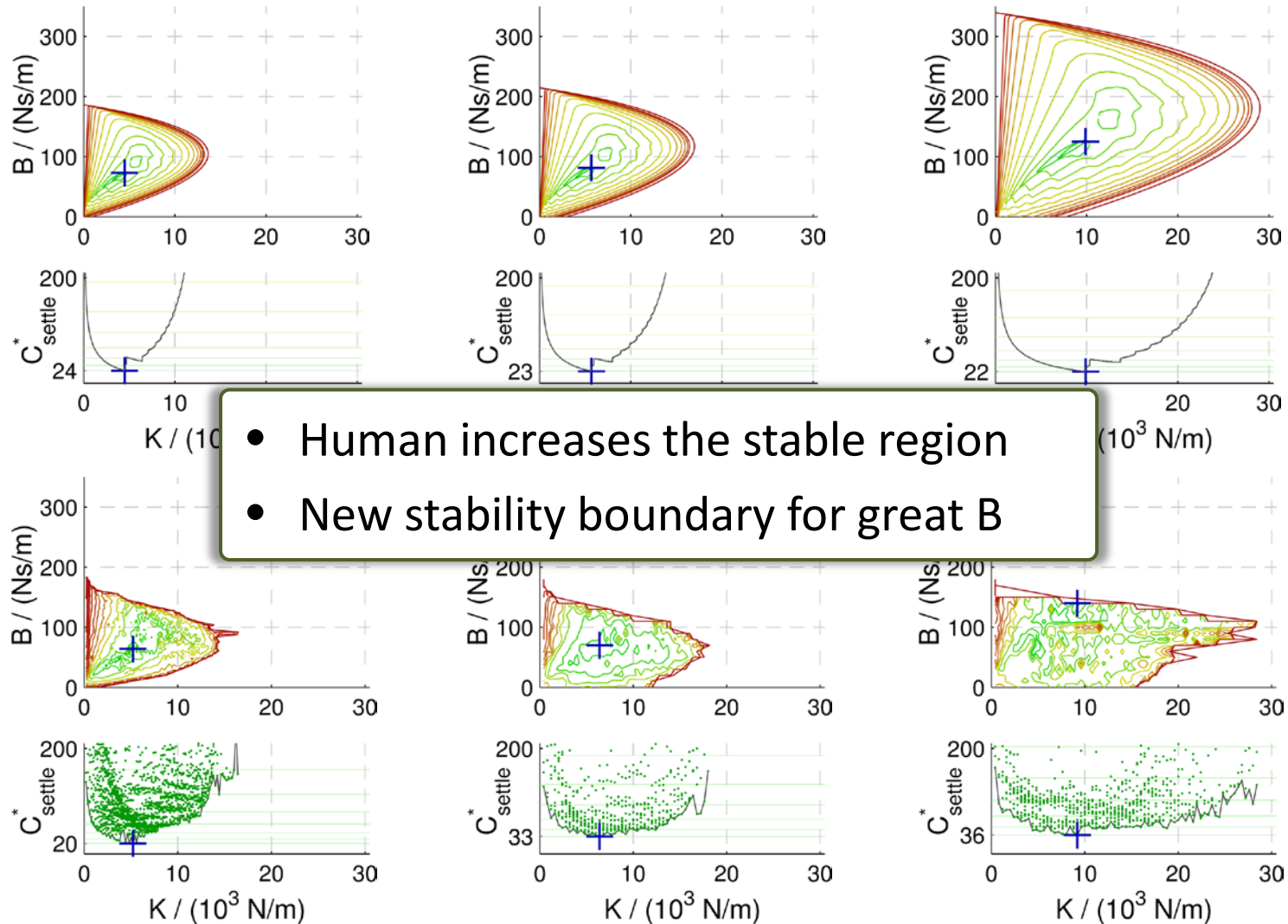
Possible Explanation [Ciáurriz 2013]:
Compliance resp. elasticities in the
mechanical structure



Experiments on the Novint Falcon

Results: 2% Settling Time

human grip	dynamic mass m		viscous damping b	
	Falcon A&B ^a	Falcon A	Falcon B	Falcon B
no operator	0.58 kg	4 Ns/m	2 Ns/m	
comfortable grip	0.65 kg	9 Ns/m	7 Ns/m	
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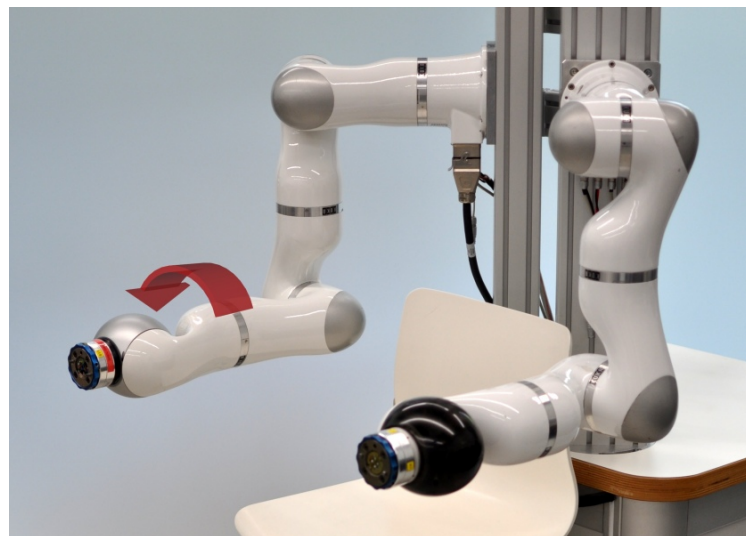


Experiments on the LWR

Experimental Procedure

A set of seven experiments:

- Influence of time delay



For each experiment a grid was defined:

$$\Delta K_{\text{rot}} = 200 \text{ Nm/rad}$$

$$\Delta B_{\text{rot}} = 1 \text{ Ns/m}$$



up to 1025 grid points

Parameter estimation results:

$$t_d = 2 \text{ ms}$$

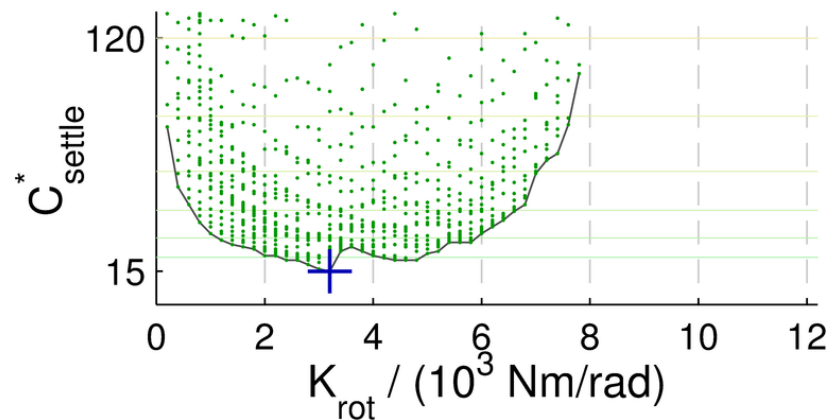
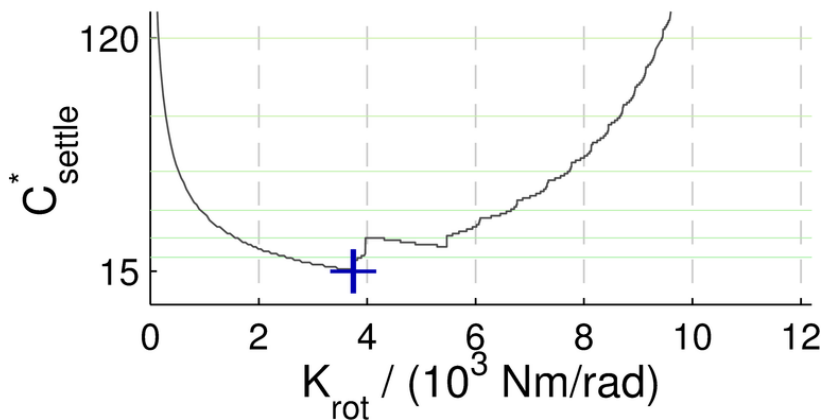
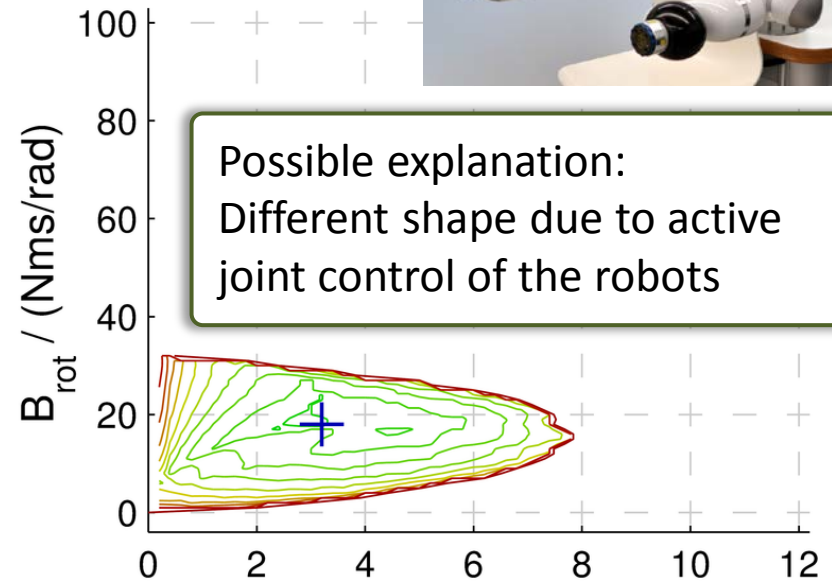
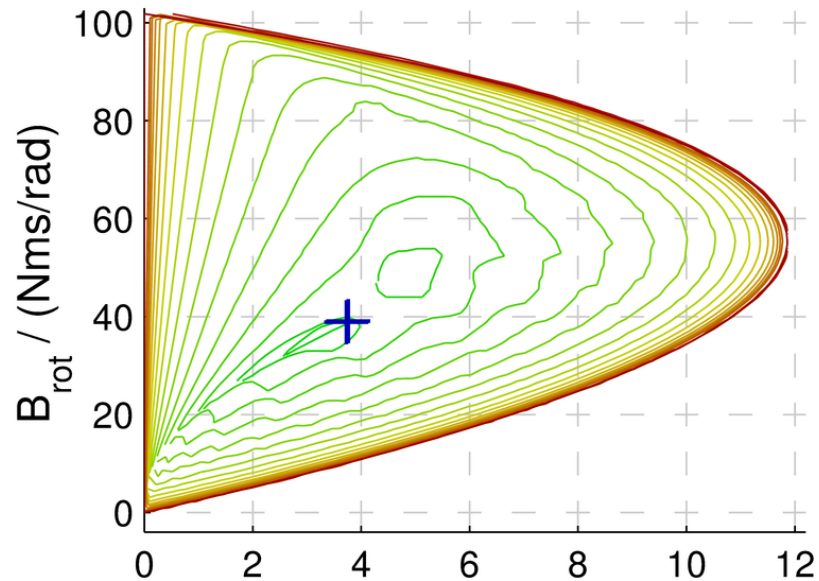
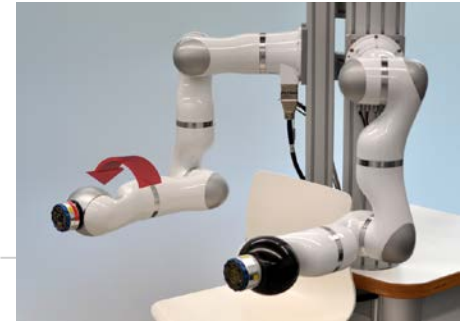
$$b_{\text{rot}} \leq 0.1 \text{ Nms/rad}$$

$$I = 0.19 \text{ kg m}^2$$



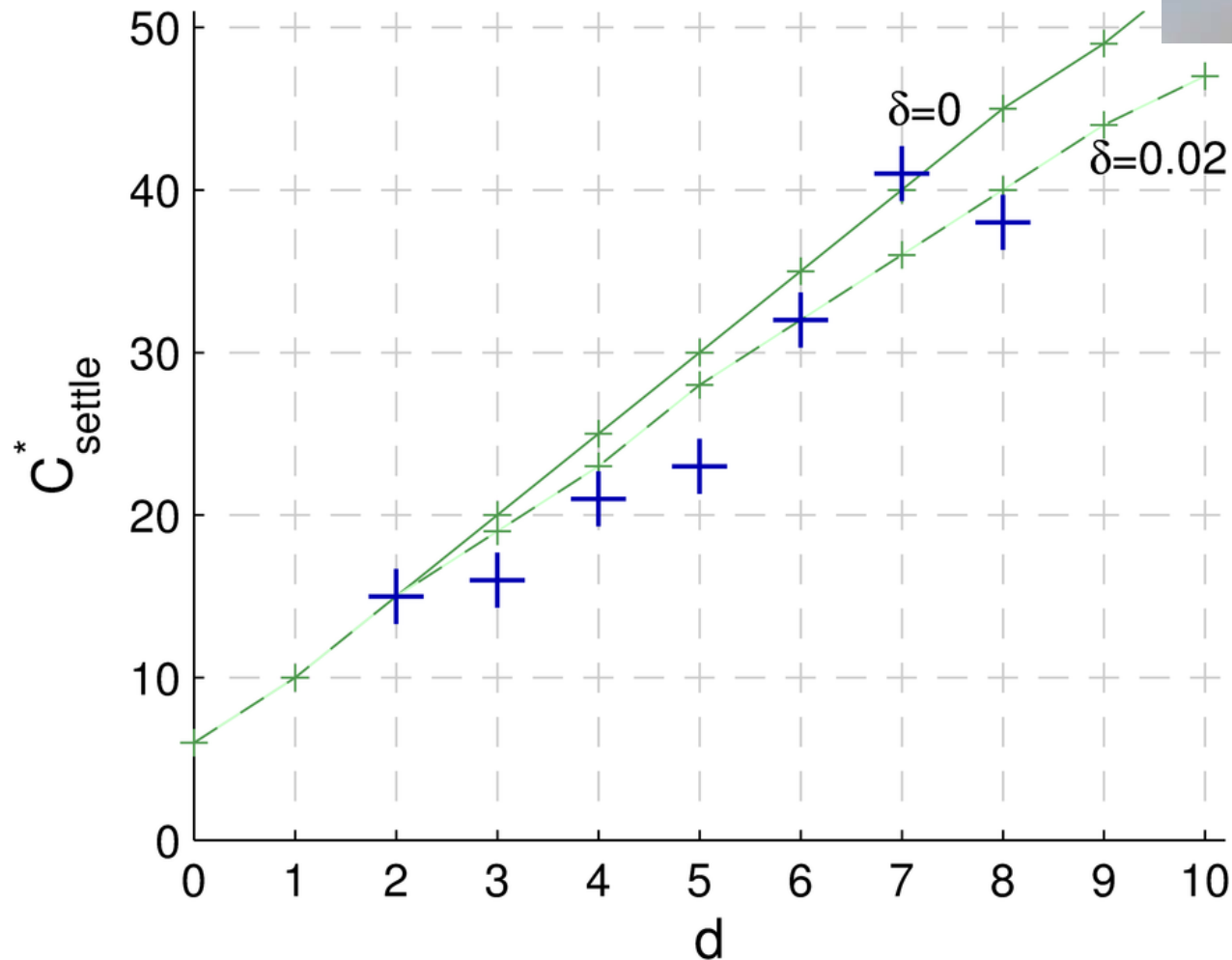
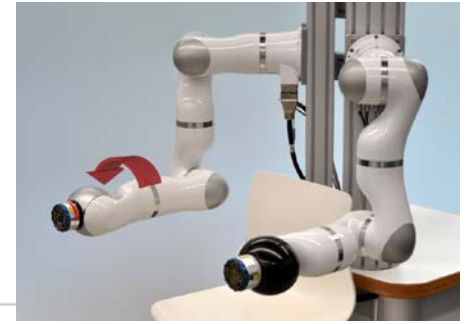
Experiments on the LWR

Results: 2% Settling Time



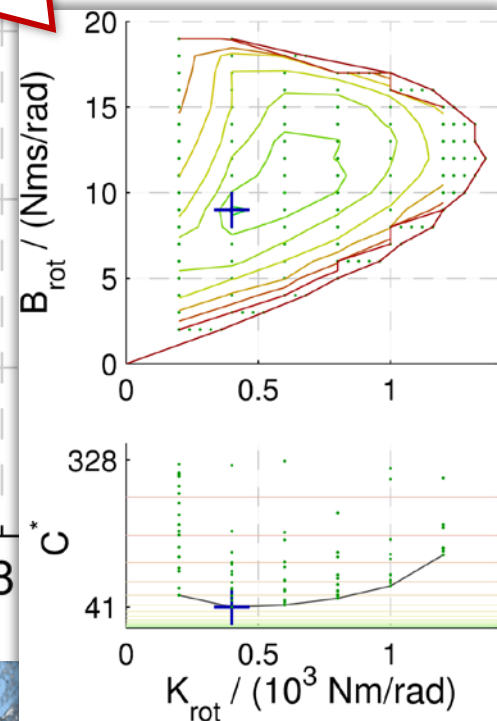
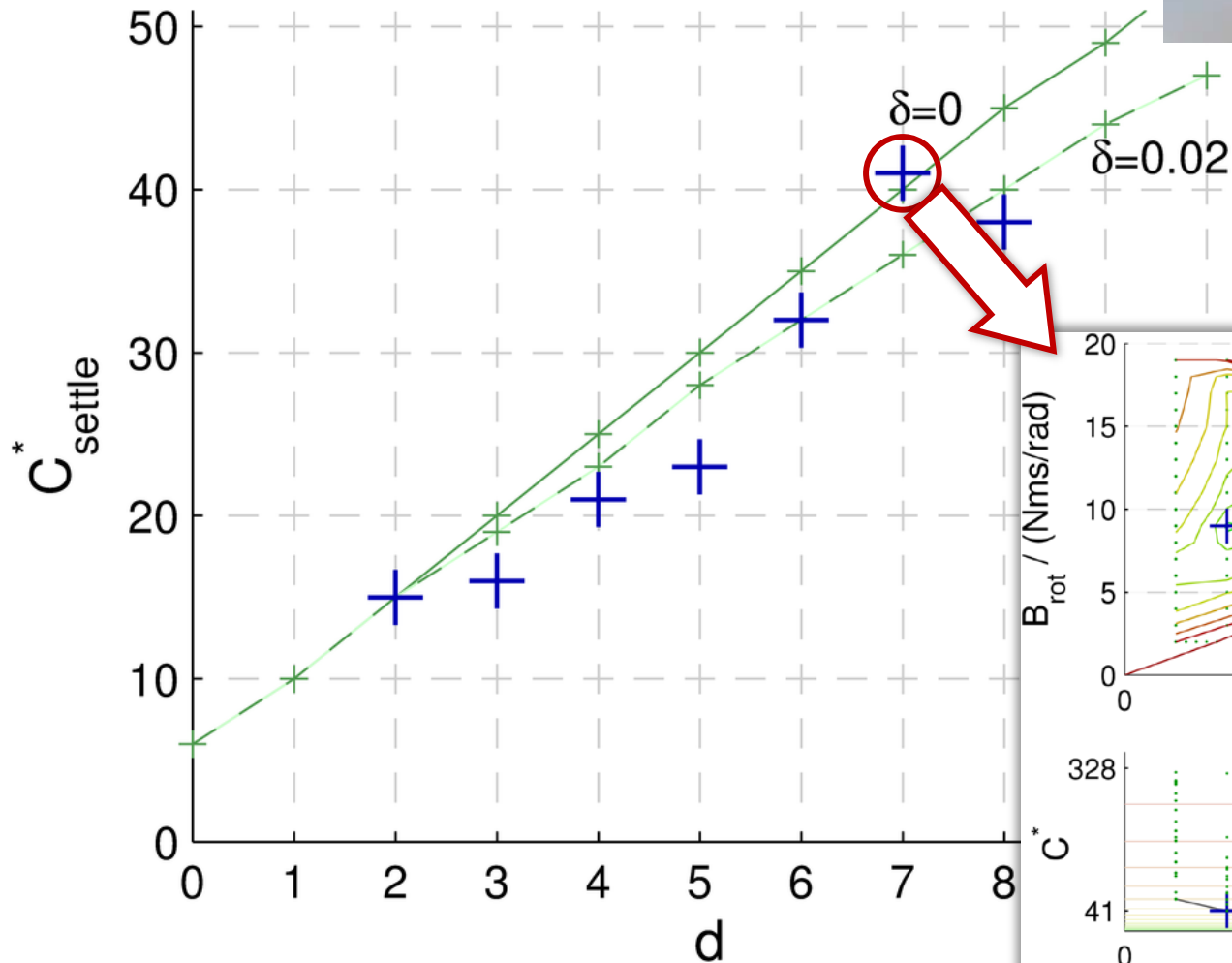
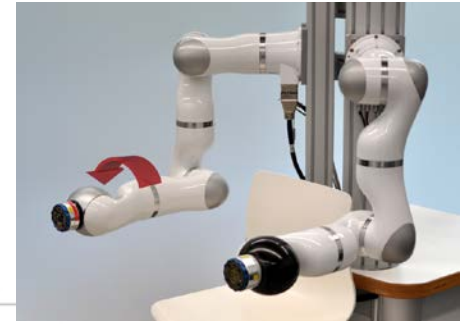
Experiments on the LWR

Results: 2% Settling Time



Experiments on the LWR

Results: 2% Settling Time



Summary

Stability Analysis

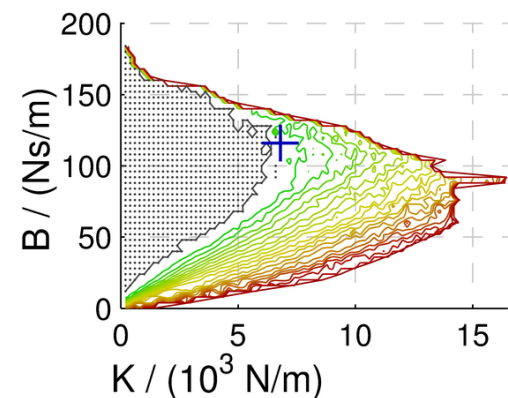
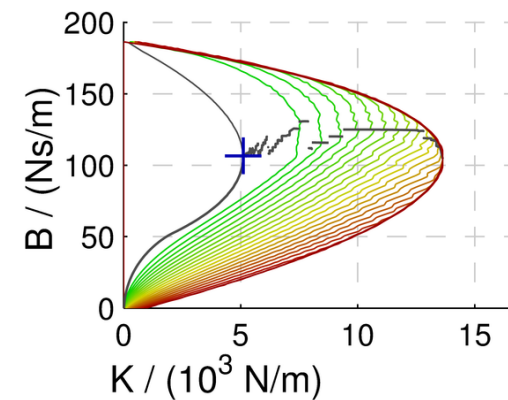
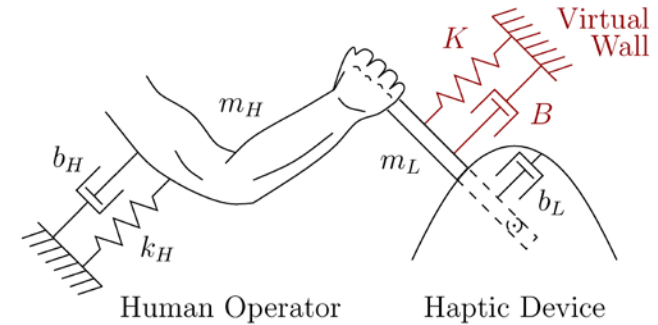
- Exact discrete-time equivalent
- Stability boundaries in normalized parameter plane
- Effect of human operator mainly by its mass contribution

Control Design

- Sub-region without overshoot inside the stable region
- Prediction function for the optimal performance (e.g. useful for cost-benefit assessment)
- Assessment and comparison of haptic devices (e.g. LWR vs. Falcon)
- (Passivity prevents optimal performance)

Experiments

- Theoretical analysis is suitable to predict the optimal performance, but not the exact shape of the boundaries
- Stabilizing effect of the human operator was confirmed



Future Work

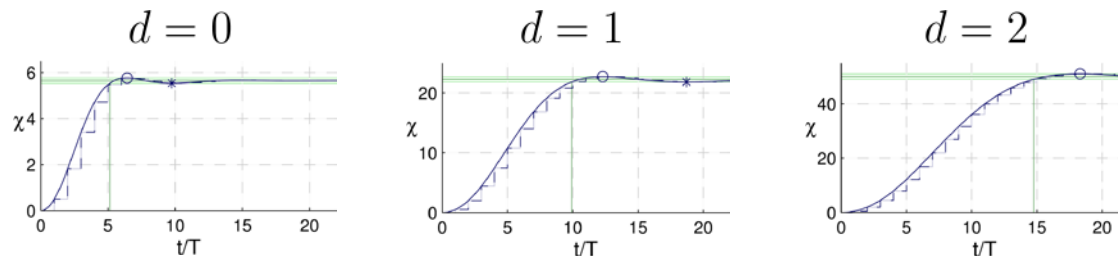
- **Analyze more comprehensive and realistic models**
 - Velocity filtering
 - Mechanical compliance
 - Nonlinearities
 - More complex virtual environments
- **Use different analysis methodologies and controllers**
 - Time-variant control
 - H-infinity
 - Lyapunov
 - Fractional-order controllers
- **Psychophysical studies**
 - Which control design approach *feels* most realistically?



Take-Home Messages

- **Rule of Thumb for the Optimal Performance**

*“Each sampling period of additional time delay causes the optimal settling time (with a 2% tolerance band) to increase by approximately **five** sampling periods.”*



- **Effect of Time-Discretization**

“Effect of discrete-time sampling corresponds to a delay of one whole sampling period T in terms of optimal cost.”

$$T_{\text{eff}} = (d + 1)T$$



Thank you for your attention!

